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# NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS

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THE COMPRESSIBLE LAMINAR BOUNDARY LAYER WITH HEAT

TRANSFER AND SMALL PRESSURE GRADIENT

By George M. Low

Lewis Flight Propulsion Laboratory Cleveland, Ohio



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#### AND SMALL PRESSURE GRADIENT

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#### SUMMARY

A perturbation method for the calculation of velocity and temperature profiles and skin-friction and heat-transfer characteristics for two-dimensional, compressible laminar boundary layers with heat transfer and a small arbitrary pressure gradient is presented. The permissible pressure gradients include those of a form and magnitude usually encountered over slender aerodynamic shapes in supersonic flight. The method applies for any constant Prandtl number, but results, aside from special examples, are presented for a Prandtl number of 0.72. For the case of heat transfer, the wall temperature is assumed constant.

A large number of universal functions are given in tabular form, so that the amount of effort required in a practical application is reduced to the arithmetic combination of several tabulated values. computation procedure is summarized in a section entitled "APPLICATION OF ANALYSIS."

The combined effects of heat transfer and pressure gradient on boundary-layer characteristics are demonstrated by applying the results of the analysis to two representative wings.

#### INTRODUCTION

Interest in the characteristics of the laminar boundary layer has increased in recent years because, under certain conditions, the boundary layer may remain laminar over large areas of airplanes and missiles. For example, Van Driest (ref. 1) has shown theoretically that if the solid boundary is cooled sufficiently, the laminar boundary layer can be stabilized regardless of Reynolds number at Mach numbers between l and 9. Sternberg (ref. 2) observed laminar boundary layers at Reynolds numbers as high as  $50 \times 10^6$  in flight tests of the V-2 rocket. Laminar boundary layers may also be expected in flight at very high altitude where the density, and hence the Reynolds number per unit length, will be low.

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Solutions of the compressible laminar-boundary-layer equations for the special case of zero pressure gradient have been obtained by several authors. The theory of Chapman and Rubesin (ref. 3), for example, presents a very simple method for calculating boundary-layer characteristics over a flat plate with arbitrary heat transfer. The more recent, and in general more exact, studies of Klunker and McLean (refs. 4 and 5), Van Driest (ref. 6), Young and Janssen (ref. 7), and Moore (ref. 8) have demonstrated that the theory of Chapman and Rubesin yields excellent results for reasonably low ambient air temperatures at Mach numbers up to about 5.

Solutions for the more general case of arbitrary heat transfer and arbitrary pressure gradient are still in an early stage of development. Tani, in a little known paper (ref. 9), used a perturbation procedure to obtain direct solutions of the boundary-layer differential equations with a Falkner-Skan type external velocity distribution ( $u_p \sim x^m$ ) and heat transfer. Results are easily obtainable from tabulated functions, but are limited to a Prandtl number of 1, small Mach numbers, and small rates of heat transfer. Furthermore, the Falkner-Skan type of external velocity distribution is not appropriate for supersonic flow over thin wings. Ginzel (ref. 10), Kalikhman (ref. 11), and Libby and Morduchow (extension of ref. 12) have obtained solutions of the compressible laminar-boundary-layer equations by an extended Pohlhausen method. However, the accuracy of the Pohlhausen method under conditions of heat transfer at high speeds has not been determined. In addition, the amount of work required in a particular application of references 10 and 11 is prohibitive because the simultaneous numerical solution of two differential equations is required. Libby and Morduchow avoid this difficulty by the additional assumption that certain variable quantities remain constant over the entire length of boundary-layer development.

The purpose of the present report is to present a method of solution developed at the NACA Lewis laboratory that is free of many of the limitations of references 9 to 12. An accurate method for calculating velocity and temperature profiles and skin-friction and heat-transfer characteristics for the compressible laminar boundary layer with heat transfer and a small pressure gradient is derived. The permissible pressure gradient may be of a form and magnitude usually encountered over thin aerodynamic shapes in supersonic flight. The solution is obtained by a method of perturbation on the flat-plate solution of Chapman and Rubesin; it constitutes the first two terms of a Maclaurin series expansion in terms of the free-stream velocity gradient parameter. The method involves the direct solution of the boundary-layer differential equations. Although the theory applies for any constant Prandtl number, tabulated results presented in this report apply, in general, for a Prandtl number of 0.72. For the case of heat transfer, results are limited to an isothermal wall.

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Solutions of the first-order perturbation equations are presented in tabular form, so that the amount of effort required in a particular application is reduced to the arithmetic combination of several tabulated values. A section of the report entitled "APPLICATION OF ANALYSIS" is included in order to facilitate the application of results in practical applications.

## ASSUMPTIONS AND LIMITATIONS

The following simplifying assumptions and limitations are imposed in addition to the usual boundary-layer assumptions:

(1) The ratio of the velocity at the outer edge of the boundary layer  $u_e$  to a reference velocity  $u_r$  can be represented by

$$\frac{u_e}{u_r} = 1 + \varepsilon \ a_N x^N \tag{1}$$

where the repeated index N indicates a summation over several values of N. (All symbols used in this report are defined in appendix A.) In equation (1) the quantity  $a_Nx^N$  represents the shape of the deviation of  $u_e$  from a constant value, while  $\epsilon$  represents the magnitude of this deviation. The quantity  $\epsilon$  is assumed small as compared with unity, whereas the quantity  $a_Nx^N$  is of normal order of magnitude. This type of external velocity distribution is capable of representing in form and magnitude those encountered over thin aerodynamic shapes at Mach numbers greater than 1.

- (2) The temperature of the solid boundary is constant under conditions of heat transfer. (Under conditions of zero heat transfer the wall temperature will be a calculated function of the pressure distribution.)
- (3) The viscosity and temperature are related linearly by the following expression:

$$\frac{\mu}{\mu_{\mathbf{r}}} = C \frac{t}{t_{\mathbf{r}}} \tag{2}$$

Chapman and Rubesin have shown that solutions of the boundary-layer equations based on equation (2) agree well with more exact solutions for flat-plate flows at Mach numbers less than 5 if the constant C is determined by matching equation (2) with Sutherland's relation at the solid boundary

$$C = \sqrt{\frac{t_w}{t_r}} \frac{(t_r + S)}{(t_w + S)}$$
 (3)

This assumption should also be reasonable for flows with slight streamwise pressure gradients, especially when the wall temperature is constant. For a nonisothermal wall an average wall temperature should be used in equation (3), as suggested in reference 3.

(4) The Prandtl number and specific heat are constant throughout the boundary layer. The restriction imposed by this assumption is not great because both Pr and  $c_p$  vary only slightly at moderate temperatures. A Prandtl number of 0.72 was used in all calculations.

#### GOVERNING EQUATIONS

<u>Differential equations and boundary conditions.</u> - The equations governing the steady laminar flow of a viscous compressible fluid in a thin boundary layer are the momentum equations

$$uu_{x} + vu_{y} = -\frac{1}{\rho} p_{x} + \frac{1}{\rho} (\mu u_{y})_{y}$$
 (4a)

$$p_{v} = 0 (4b)$$

the equation of continuity

$$(\rho u)_{x} + (\rho v)_{v} = 0$$
 (5)

the energy equation

$$\rho c_{p}(ut_{x} + vt_{y}) = up_{x} + (kt_{y})_{y} + \mu(u_{y})^{2}$$
(6)

and the equation of state

$$p = \rho Rt \tag{7}$$

The following boundary conditions are imposed on the momentum and energy equations:

$$u(x,0) = 0 u(x,\infty) = u_e$$

$$v(x,0) = 0 t(x,\infty) = t_e$$

$$t(x,0) = t_w (heat transfer)$$

$$t_v(x,0) = 0 (zero heat transfer)$$
(8)

At the outer edge of the boundary layer, velocity and pressure are related by the Bernoulli equation, which is

$$\frac{\mathrm{d}p_{\mathrm{e}}}{\mathrm{d}x} = -\rho_{\mathrm{e}}u_{\mathrm{e}} \frac{\mathrm{d}u_{\mathrm{e}}}{\mathrm{d}x} \tag{9}$$

The energy equation that applies at the outer edge of the boundary layer is

$$c_p T = c_p t_e + \frac{u_e^2}{2}$$
 (10)

Transformation of Howarth. - In reference 13 Howarth introduced a transformation which, when applied to the momentum and energy equations, yields equations similar in form to the incompressible-boundary-layer equations. First, it is convenient to introduce the dimensionless variables

$$p^{*} = p/p_{r} u^{*} = u/u_{r}$$

$$t^{*} = t/t_{r} v^{*} = v/u_{r}$$

$$\rho^{*} = \rho/\rho_{r} \mu^{*} = \mu/\mu_{r}$$
(11)

Howarth's transformation proceeds as follows: The independent variables x and y are related to the variables x and y according to the following transformation:

$$x \equiv x$$

$$n \equiv \sqrt{\frac{p^*}{C}} \int_0^y \frac{1}{t^*} dy$$
(12)

where n distorts the scale in the direction normal to the surface. The derivatives with respect to x and y can be expressed as

$$\frac{\partial}{\partial y} = \sqrt{\frac{c}{p^*}} \frac{1}{1} \frac{\partial}{\partial n}$$

$$(13)$$

Equation (5) is satisfied by a stream function defined as follows:

$$\rho^* u^* = \psi_y$$
$$\rho^* v^* = - \psi_x$$

The stream function  $\psi(x,y)$  can be related to a transformed function  $\phi(x,n)$  by

$$\psi(x,y) \equiv \sqrt{Cp^*} \phi(x,n) \tag{14}$$

The velocity components now become

$$v = -\frac{u_r \cdot \sqrt{Cp^*}}{o^*} \left( \phi_x + \phi_n \cdot n_x + \frac{1}{2} \cdot \frac{p_x^*}{p^*} \cdot \phi \right)$$
(15)

Substitution of equations (2), (7), (9), and (12) to (15) into equation (4a) yields the momentum equation in the transformed (x,n) plane

$$\phi_{n} \phi_{nx} - \phi_{x} \phi_{nn} - \frac{v_{r}}{u_{r}} \phi_{nnn} = \frac{u_{e}^{*}}{t_{e}^{*}} \frac{du_{e}^{*}}{dx} \left[ t^{*} - \frac{1}{2} \gamma M_{r}^{2} \phi \phi_{nn} \right]$$
(16)

The energy equation in the x,n-plane becomes

The following boundary conditions apply to equations (16) and (17):

$$\phi(x,0) = 0 \qquad \qquad \phi_n(x,\infty) = u_e^*$$

$$\phi_n(x,0) = 0$$

$$t^*(x,0) = t_w^* \text{ (heat transfer)}$$

$$t_n^*(x,0) = 0 \text{ (zero heat transfer)}$$

$$t^*(x,\infty) = t_e^*$$

#### PERTURBATION ANALYSIS

Expansion of momentum and energy equations in powers of  $\varepsilon$ . - For the special case of  $u_{\xi}^*=1$  (zero pressure gradient), equations (16) and (17) become identical to the momentum and energy equations solved by Chapman and Rubesin. It therefore appears logical to let  $u_{\varepsilon}^*$  differ from unity by a small amount in order to obtain a perturbation solution for flows with small pressure gradients. As discussed under ASSUMPTIONS AND LIMITATIONS, the external velocity at the outer edge of the boundary layer is taken to be of the following form:

$$u_e^* = 1 + \varepsilon a_N x^N \tag{1}$$

Substitution of equations (1) and (7) into equations (9) and (10) and elimination of higher-order terms yield:

$$p^* = 1 - \gamma M_r^2 \epsilon a_N x^N$$
 (18)

and

$$t_{e}^{*} = 1 - (\gamma - 1) M_{r}^{2} \varepsilon a_{N} x^{N}$$
 (19)

Within the boundary layer the stream function and temperature are replaced by their Maclaurin series expansions in terms of the velocity gradient parameter  $\ensuremath{\epsilon}$ :

$$\varphi(\mathbf{x},\mathbf{n},\varepsilon) = \overline{\varphi}(\mathbf{x},\mathbf{n}) + \varepsilon \, \mathbf{a}_{\overline{N}} \, \overline{\overline{\varphi}}_{\overline{N}}(\mathbf{x},\mathbf{n}) + \varepsilon^2 \, \mathbf{a}_{\overline{NM}} \, \overline{\overline{\varphi}}_{\overline{NM}}(\mathbf{x},\mathbf{n}) + \dots$$
 (20)

$$t^*(x,n,\epsilon) = \overline{t}(x,n) + \epsilon a_N \overline{t}_N(x,n) + \epsilon^2 a_{NM} \overline{t}_{NM}(x,n) + \dots \qquad (21)$$

A sequence of momentum and energy equations is obtained by substitution of equations (1) and (18) to (21) into equations (16) and (17), and by equating coefficients of like powers of  $\varepsilon$ . The zero-order equations, obtained by equating coefficients of  $(\varepsilon)^0$ , are:

$$\overline{\phi}_{n} \overline{\phi}_{nx} - \overline{\phi}_{x} \overline{\phi}_{nn} - \frac{v_{r}}{u_{r}} \overline{\phi}_{nnn} = 0$$
 (22)

$$\overline{\varphi}(x,0) = \overline{\varphi}_n(x,0) = 0 \qquad \overline{\varphi}_n(x,\infty) = 1$$

and

$$\overline{\varphi}_{n} \overline{t}_{x} - \overline{\varphi}_{x} \overline{t}_{n} - \frac{v_{r}}{u_{r} Pr} \overline{t}_{nn} = (\gamma - 1) \frac{v_{r} M_{r}^{2}}{u_{r}} (\overline{\varphi}_{nn})^{2}$$
(23)

$$\overline{t}(x,0) = t_W^* \text{ (heat transfer)}$$

$$\overline{t}_n(x,0) = 0$$
 (zero heat transfer)

$$\overline{t}(x,\infty) = 1$$

Equating coefficients of  $\epsilon$  yields the first-order equations:

$$\overline{\phi}_{n} \ \overline{\phi}_{Nnx} + \overline{\phi}_{nx} \ \overline{\phi}_{Nn} - \overline{\phi}_{x} \ \overline{\phi}_{Nnn} - \overline{\phi}_{nn} \ \overline{\phi}_{Nx} - \frac{v_{r}}{u_{r}} \ \overline{\phi}_{Nnnn}$$

$$= N \ x^{N-1} \ (\overline{t} - \frac{r}{2} M_{r}^{2} \ \overline{\phi} \ \overline{\phi}_{nn}) \ (24)$$

$$\vec{\Phi}_{N}(x,0) = \vec{\Phi}_{Nn}(x,0) = 0$$
  $\vec{\Phi}_{Nn}(x,\infty) = x^{N}$ 

and

$$\overline{\phi}_{Nn} \ \overline{t}_{x} + \overline{\phi}_{n} \ \overline{t}_{Nx} - \overline{\phi}_{Nx} \ \overline{t}_{n} - \overline{\phi}_{x} \ \overline{t}_{Nn} - \frac{\nu_{r}}{u_{r} \ \overline{Pr}} \ \overline{t}_{Nnn}$$

$$= 2 \frac{\nu_{r}}{u_{r}} (\gamma - 1) M_{r}^{2} \ \overline{\phi}_{nn} \ \overline{\phi}_{Nnn} - Nx^{N-1} M_{r}^{2} \left[ \frac{\gamma}{2} \ \overline{\phi} \ \overline{t}_{n} + (\gamma - 1) \ \overline{t} \ \overline{\phi}_{n} \right]$$

$$\overline{t}_{N}(x, 0) = 0 \quad \text{or} \quad \overline{t}_{Nn}(x, 0) = 0 \qquad \overline{t}_{N}(x, \infty) = - (\gamma - 1)x^{N} M_{r}^{2}$$

$$(25)$$

The higher-order equations are obtained by equating coefficients of  $\epsilon^2$ ,  $\epsilon^3$ , and so forth. If it is assumed that the functions  $\overline{\phi}$ ,  $\overline{\phi}$ ,

and so forth, and the functions  $\bar{t}$ ,  $\bar{t}$ ,  $\bar{t}$ , and so forth, are of the same order of magnitude, then, since  $\epsilon$  is postulated to be small, all additional contributions will be of second or higher order and hence may be neglected in a first-order treatment. Further justification for neglecting the higher-order equations comes from reference 14, where it is shown that for incompressible flow the function  $\bar{\phi}$  is numerically much smaller than  $\bar{\phi}$  and  $\bar{\phi}$ . The second-order terms therefore should contribute very little to the solution of the boundary-layer equations for flows with small pressure gradients.

The permissible magnitude of the pressure gradient depends largely on the length of run over which it acts. For example, a very small pressure gradient can cause laminar separation if it acts over a large distance. The present method can be applied only if all the deviations in boundary-layer characteristics caused by the pressure gradient are small.

The only dependent variable appearing in equation (22) is  $\overline{\phi}$ , so that the solution of this equation is independent of all following equations. Furthermore, each succeeding equation involves only one new dependent variable, so that each equation can be solved in principle once the preceding equations have been solved. Equations (22) to (25) still contain two independent variables, however, and require further reduction to make them amenable to solution.

Solution of zero-order equation. - The zero-order equations may be transformed to ordinary differential equations by introduction of the Blasius characteristic variable  $\eta$ :

$$\eta \equiv \frac{n}{2} \sqrt{\frac{u_r}{v_r x}} \tag{26}$$

The stream function  $\overline{\phi}(x,n)$  is related to a new function  $f(\eta)$  as follows:

$$\overline{\phi}(x,n) = \sqrt{\frac{v_r x}{u_r}} f(\eta)$$
 (27)

The temperature in the x,n-coordinate system is equal to the temperature in the  $\eta$ -system

$$\overline{t}(x,n) = \overline{t}(\eta) \tag{28}$$

With the aid of equations (26), (27), and (28), equations (22) and (23) can be written

$$f''' + ff'' = 0 \tag{29}$$

$$\overline{t}" + Pr f \overline{t}' = - (Pr) \left(\frac{\gamma - 1}{4}\right) M_r^2 (f")^2$$
(30)

where the primes indicate differentiation with respect to  $\eta$ . The boundary conditions of f and t are

$$f(0) = 0 f'(\infty) = 2$$

$$f'(0) = 0$$

$$\overline{t}(0) = t_W^* or \overline{t}'(0) = 0$$

$$\overline{t}(\infty) = 1$$

Equation (29) is the well-known Blasius equation which has been solved by several investigators. In order to eliminate  $M_{r}^{2}$  as a parameter in the solution of equation (30), this equation is split into two parts in the following manner:

$$\overline{t}(\eta) = 1 + \frac{\gamma - 1}{2} M_r^2 r(\eta) + K s(\eta)$$
 (31)

where  $r(\eta)$  and  $s(\eta)$  satisfy the following equations:

$$r'' + Pr f r' = -\frac{Pr}{2} (f'')^2$$
 (32)

$$s'' + Pr f s' = 0$$
 (33)

The following boundary conditions are applied to equations (32) and (33):

$$r'(0) = 0$$
  $r(\infty) = 0$   
 $s'(0) = - \lceil f''(0) \rceil^{Pr}$   $s(\infty) = 0$ 

Although the solution of equations (32) and (33) can be written in terms of quadratures, as shown in reference 3, the numerical solution of the differential equations was found more convenient. Numerical solutions, as discussed in appendix B, were made by Lynn U. Albers.

The functions f''(0), r(0), s(0), and s'(0) are listed in table I; all other functions resulting from the solution of the zero-order equations can be found in table II. The constant K (eq. (31)) is related to the rate of heat transfer and hence to the wall temperature. Its value is determined by solving equation (31) at  $\eta = 0$ .

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$$K = \frac{1}{s(0)} \left[ (\overline{t}_{w} - 1) - \frac{\gamma - 1}{2} M_{r}^{2} r(0) \right]$$
 (34)

where the wall temperature ratio  $\overline{t}_w = \frac{t_w}{t_r}$  is, in general, prescribed. For the case of zero heat transfer, K vanishes.

Solution of first-order equations. - The first-order momentum and energy equations can also be transformed to ordinary differential equations with the aid of the Blasius variable together with the following functions:

$$\stackrel{=}{\phi_{N}}(x,n) = 2 \sqrt{\frac{v_{r} \times x}{u_{r}}} x^{N} g_{N}(\eta)$$
 (35)

and

$$t_{N}(x,n) = - (\gamma-1) M_{r}^{2} x^{N} h_{N}(\eta) \quad (zero heat transfer)$$
 (36)

$$\overline{\overline{t}}_{N}(x,n) = - (\gamma-1) M_{r}^{2} x^{N} H_{N}(\eta) \quad \text{(heat transfer)}$$
 (37)

Equations (24) and (25) are now written

$$g_N^{""} + f g_N^{"} - 2N f' g_N^{"} + (2N + 1) f'' g_N$$

$$= -4N \left\{ 1 + M_T^2 \left[ \left( \frac{\gamma - 1}{2} \right) r - \frac{\gamma}{8} f f'' \right] + Ks \right\}$$
(38)

and

$$h_N^{"}$$
 + Pr  $fh_N^{"}$  - 2Pr  $Nf^{"}h_N$ 

$$= \Pr \left[ \frac{4N+2}{(\gamma-1)M_{r}^{2}} g_{N} \overline{t}' + f'' g_{N}'' - \frac{\gamma}{\gamma-1} N f \overline{t}' - 2Nf' \overline{t} \right]$$
(39)

The function  $H_N(\eta)$  satisfies the same equation as is satisfied by  $h_N(\eta)$  (eq. (41)), but is subject to different boundary conditions. The boundary conditions are

$$\begin{split} g_{\overline{N}}(0) &= 0 & g_{\overline{N}}^{i}(\infty) = 1 \\ g_{\overline{N}}^{i}(0) &= 0 & h_{\overline{N}}(\infty) = 1 & (\text{zero heat transfer}) \\ h_{\overline{N}}(0) &= 0 & h_{\overline{N}}(\infty) = 1 & (\text{heat transfer}) \end{split}$$

The solution of equation (38) can be obtained in closed form for the special cases of  $N=-\frac{1}{2}$  (ref. 15) and N=0 (see appendix C). In the general case, however, the equation was solved numerically. In order to obtain a numerical solution which applies over a range of Mach numbers and heat-transfer rates, the parameters  $M_r$  and K were eliminated from equation (38) by splitting the function g into a linear combination of three independent functions:

$$g_N = g_{N1} + M_T^2 g_{N2} + K g_{N3}$$
 (40)

where the three new functions satisfy the following equations:

$$g_{N1}^{""} + fg_{N1}^{"} - 2Nf'g_{N1}^{"} + (2N + 1) f"g_{N1}^{"} = -4N$$
 (41)

$$g_{N2}^{""} + fg_{N2}^{"} - 2Nf'g_{N2}^{"} + (2N + 1) f''g_{N2} = \frac{N}{2} [\gamma ff'' - 4(\gamma - 1)r]$$
 (42)

$$g_{N3}^{""} + fg_{N3}^{"} - 2Nf'g_{N3}^{"} + (2N + 1) f"g_{N3} = -4Ns$$
 (43)

$$g_{Ni}(0) = g_{Ni}(0) = 0$$
 (i = 1,2,3)

$$g_{N1}^{\dagger}(\infty) = 1$$
  $g_{N2}^{\dagger}(\infty) = g_{N3}^{\dagger}(\infty) = 0$ 

The first-order energy equation can be solved in closed form for  $N=-\frac{1}{2}$  (ref. 15) and N=0 (appendix C). For zero heat transfer, a closed form solution of equation (39) for Pr=1 can also be obtained for all values of N, as shown in appendix D. For other cases the solution of equation (39) is again found numerically after several parameters have been eliminated by replacing the equation by the following system:

$$h_{N} = h_{NL} + M_{r}^{2} h_{N2}$$
 (44)

where

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$$h_{N1}^{"}$$
 + Pr  $fh_{N1}^{"}$  - 2Pr  $Nf'h_{N1}$  = Pr  $\left[ (2N + 1)g_{N1}r' + f''g_{N1}'' - 2Nf' \right]$  (45)

$$= \Pr \left[ (2N + 1)g_{N2} r' + f''g_{N2}'' - \frac{\gamma Nr'f}{2} - N(\gamma - 1)rf' \right]$$
 (46)

$$h_{N1}^{t}(0) = 0$$
  $h_{N1}(\infty) = 1$   $h_{N2}(\infty) = 0$  (i = 1,2)

Equations (44) to (46) apply for the case of zero heat transfer. For flows with heat transfer, the following equations are obtained:

$$H_{N} = H_{NL} + M_{r}^{2} H_{N2} + KH_{N3} + \frac{K}{M_{r}^{2}} H_{N4} + \frac{K^{2}}{M_{r}^{2}} H_{N5}$$
 (47)

The functions  $H_{N1}(\eta)$  and  $H_{N2}(\eta)$  satisfy the same equations or are satisfied by  $h_{N1}(\eta)$  and  $h_{N2}(\eta)$ , but are subject to modified boundary conditions. The remaining functions in equation (47) satisfy the following equations:

$$H_{N3}^{"} + Pr fH_{N3}^{"} - 2Pr Nf'H_{N3} = Pr \left[ (2N + 1) \left( \frac{2g_{N2}s'}{\gamma - 1} + r' g_{N3} \right) + f''g_{N3}^{"} - \frac{\gamma Nfs'}{\gamma - 1} - 2Nf's \right]$$

$$H_{N4}^{"} + Pr fH_{N4}^{"} - 2Pr Nf'H_{N4} = Pr \left[ (2N + 1) \left( \frac{2g_{N1}s'}{\gamma - 1} \right) \right]$$

$$H_{N5}^{"} + Pr fH_{N5}^{"} - 2Pr Nf'H_{N5} = Pr \left[ (2N + 1) \left( \frac{2g_{N3}s'}{\gamma - 1} \right) \right]$$

$$H_{N1}^{"}(0) = 0 \qquad H_{N1}^{"}(\infty) = 1 \qquad (i = 1, 2, 3, 4, 5)$$

$$(48)$$

$$H_{N2}(\infty) = H_{N3}(\infty) = H_{N4}(\infty) = H_{N5}(\infty) = 0$$

The elimination of  $M_{r}$  and K as parameters in the first-order equations has thus yielded ten equations for each value of N. These

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equations were solved numerically for N=1, 2, and 3, as discussed in appendix B. The functions  $g_N^i$ , which represent the first-order velocity corrections, and  $h_N$  and  $H_N$ , which represent the first-order temperature corrections, are presented graphically in figures 1, 2, and 3. Tabulated results of all solutions of the first-order equations can be found in tables III, IV, and V. Initial values are also tabulated in table I.

The solutions of the zero- and first-order equations can now be combined to yield velocity and temperature profiles, skin-friction and heat-transfer coefficients, recovery factors, and displacement thickness.

#### BOUNDARY-LAYER CHARACTERISTICS

Velocity and temperature profiles. - The dimensionless velocity  $u^*$  is related to the characteristic variable  $\eta$  through equations (15), (20), (27), and (35) in the following manner:

$$u^* = \frac{1}{2} f'(\eta) + \epsilon a_N x^N g_N'(\eta)$$
 (51)

From equations (21), (28), (31), and (36) or (37), the temperature profile can be expressed in terms of  $\eta$  as

$$t^* \cong 1 + \frac{\gamma - 1}{2} M_r^2 \left[ r(\eta) - 2\varepsilon \, a_N^2 x^N h_N(\eta) \right]$$
 (52)

for the case of zero heat transfer. For flows with arbitrary heat transfer, the following expression applies:

$$t^* \cong 1 + K s(\eta) + \frac{\gamma - 1}{2} M_r^2 \left[ r(\eta) - 2\varepsilon a_N x^N H_N(\eta) \right]$$
 (53)

Equations (51), (52), and (53) represent the velocity and temperature profiles as functions of  $\eta$ . The transformation to the physical (x,y-) plane, according to equations (12) and (26), is

$$y = 2 \sqrt{\frac{v_r xC}{p^* u_r}} \int_{O}^{\eta} t^* d\eta$$
 (54)

The value of  $t^*$  can be obtained from equations (52) or (53), while  $p^*$  is given by equation (18). Thus, for zero heat transfer,

$$y \approx 2 \sqrt{\frac{v_r \times C}{u_r}} \left\{ \left[ 1 + \frac{\gamma}{2} M_r^2 \epsilon a_N x^N \right] \left[ \eta + \frac{\gamma - 1}{2} M_r^2 \operatorname{Ir}(\eta) \right] - (\gamma - 1) M_r^2 \epsilon a_N x^N \operatorname{Ih}_N \right\}$$
(55)

and for arbitrary rates of heat transfer,

$$y \approx 2 \sqrt{\frac{v_{r} \times C}{u_{r}}} \left\{ \left[ 1 + \frac{\gamma}{2} M_{r}^{2} \epsilon a_{N} x^{N} \right] \left[ \eta + \frac{\gamma - 1}{2} M_{r}^{2} \operatorname{Ir}(\eta) + KI s(\eta) \right] - (\gamma - 1) \epsilon a_{N} x^{N} M_{r}^{2} \operatorname{IH}_{N}(\eta) \right\}$$
(56)

Skin friction and heat transfer. - The shearing stress at a point in the boundary layer can be obtained from equations (2), (13), and (15):

$$\tau \equiv \mu \frac{\partial u}{\partial y} = \mu_r u_r \sqrt{c_p} \phi_{nn}$$
 (57)

In terms of the Blasius variable  $\eta$  and after substitution of equation (18) for  $p^*$ , equation (57) becomes

$$\tau \cong \frac{\mathbf{u_r}}{4} \sqrt{\frac{\mu_r \rho_r \mathbf{u_r} C}{\mathbf{x}}} \left\{ \mathbf{f}''(\eta) + 2 \epsilon \mathbf{a_N} \mathbf{x}^N \left[ \mathbf{g_N''}(\eta) - \frac{\gamma}{4} \mathbf{M_r^2} \mathbf{f''}(\eta) \right] \right\}$$
 (58)

Local and average skin-friction coefficients are obtained from the wall shearing stress and the following respective definitions:

$$c_{\mathbf{f}} = \frac{\tau_{\mathbf{w}}}{\frac{1}{2} \rho_{\mathbf{r}} u_{\mathbf{r}}^2} \tag{59}$$

and

$$C_{\mathbf{F}} = \frac{1}{\frac{1}{2} \rho_{\mathbf{r}} u_{\mathbf{r}}^2 x} \int_{0}^{x} \tau_{\mathbf{w}} dx$$
 (60)

A local skin-friction parameter is obtained from equations (58) and (59):

$$c_{f}\sqrt{\frac{\operatorname{Re}}{C}} \approx \frac{1}{2} \left\{ f''(0) + 2\varepsilon \, a_{N}x^{N} \left[ g_{N}''(0) - \frac{\gamma}{4} \, M_{r}^{2} \, f''(0) \right] \right\}$$
 (61)

The average friction drag parameter, obtained from equations (58) and (60), is

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$$C_{\mathbf{F}}\sqrt{\frac{\mathrm{Re}}{\mathrm{C}}} \cong f''(0) + \frac{2\varepsilon \, a_{\mathbf{N}}x^{\mathbf{N}}}{2\mathbf{N} + 1} \left[ g_{\mathbf{N}}''(0) - \frac{\gamma}{4} \, M_{\mathbf{r}}^{2} \, f''(0) \right] \tag{62}$$

The local rate of heat transfer from the surface is given by

$$q \equiv -k \frac{\partial t}{\partial y}\Big|_{W} = -\frac{c_{\underline{p}} t_{\underline{r}} \mu_{\underline{r}}}{Pr} \sqrt{Cp^{*}} t_{\underline{n}}^{*}\Big|_{W}$$

$$\equiv -\frac{c_{\underline{p}}}{2 Pr} t_{\underline{r}} \sqrt{\frac{\mu_{\underline{r}} \rho_{\underline{r}} u_{\underline{r}} C}{x}} \left\{ K s'(0) - (\gamma-1) M_{\underline{r}}^{2} \epsilon a_{\underline{N}} x^{\underline{N}} \left[ H_{\underline{N}}^{i}(0) + \frac{\gamma}{2(\gamma-1)} K s'(0) \right] \right\}$$
(63)

A dimensionless heat-transfer parameter can now be written as follows:

$$\frac{Nu}{\sqrt{C \operatorname{Re}}} \cong \frac{1}{2(\dot{t}_{aw}^{*} - \dot{t}_{w}^{*})} \left\{ \operatorname{K} s'(0) - (\gamma-1) \operatorname{M}_{r}^{2} \varepsilon a_{N} x^{N} \left[ \operatorname{H}_{N}^{t}(0) + \frac{\gamma}{2(\gamma-1)} \operatorname{K} s'(0) \right] \right\}$$
(64)

where the dimensionless adiabatic wall temperature is

$$t_{aw}^* \approx 1 + \frac{\gamma - 1}{2} M_r^2 \left[ r(0) - 2\varepsilon a_N x^N h_N(0) \right]$$
 (65)

Temperature recovery factor. - The temperature recovery factor is derived from equations (19) and (65):

$$F_{R} = \frac{t_{aw}^{*} - t_{e}^{*}}{T^{*} - t_{e}^{*}} \approx r(0) + 2\epsilon a_{N}x^{N} \left[1 - h_{N}(0) - r(0)\right]$$
 (66)

It is evident from the computed results that  $h_{\rm Nl}(0)$  varies very little with N and is approximately equal to 1-r(0). With the aid of equation (44), equation (66) is therefore reduced to the following:

$$F_{R} \cong r(0) - 2\varepsilon a_{N} x^{N} M_{r}^{2} h_{N2}(0)$$
 (67)

. <u>Displacement thickness</u>. - The boundary-layer displacement thickness is, by definition,

$$\delta^* \equiv \int_0^\infty \left( 1 - \frac{\rho u}{\rho_e u_e} \right) dy \tag{68}$$

$$= 2 \sqrt{\frac{c v_r x}{p^* u_r}} \int_{0}^{\infty} \left(t^* - \frac{t_e^*}{u_e^*} u^*\right) d\eta$$
 (69)

With the appropriate expressions for  $p^*$ ,  $t^*$ , and  $u^*$ , equation (69) becomes, for flows with heat transfer,

$$\delta^* \cong 2 \sqrt{\frac{c v_r x}{u_r}} \int_0^{\infty} \left\{ \left[ 1 + \frac{\gamma}{2} M_r^2 \epsilon a_N x^N \right] \left[ \left( 1 - \frac{1}{2} f^{\dagger} \right) + Ks + \frac{\gamma - 1}{2} M_r^2 r \right] + \epsilon a_N x^N \left[ \frac{1}{2} f^{\dagger} - g_N^{\dagger} \right] + \frac{\gamma - 1}{2} M_r^2 \epsilon a_N x^N \left[ f^{\dagger} - 2H_N \right] \right\} d\eta$$
 (70)

Integration of equation (70) yields, for Pr = 0.72,

$$\delta^* \cong \sqrt{\frac{C \nu_r x}{u_r}} \left\{ \left[ 1 + \frac{\gamma}{2} M_r^2 \epsilon a_N x^N \right] \left[ 1.7208 + 4.0218 K + 1.1094 (\gamma-1) M_r^2 \right] + \epsilon a_N x^N \left[ \alpha_N + (\gamma-1) M_r^2 B_N \right] \right\}$$
(71)

For the case of zero heat transfer K will vanish and  $B_{\mathbb{N}}$  is replaced by  $\beta_{\mathbb{N}}.$  The relation between the functions appearing in equation (71) and the functions tabulated in table VI is:

$$\alpha_{N} = \alpha_{N1} + M_{r}^{2} \alpha_{N2} + K \alpha_{N3}$$

$$\beta_{N} = \beta_{N1} + M_{r}^{2} \beta_{N2}$$

$$\beta_{N} = \beta_{N1} + M_{r}^{2} \beta_{N2} + K \beta_{N3} + \frac{K}{M_{r}^{2}} \beta_{N4} + \frac{K^{2}}{M_{r}^{2}} \beta_{N5}$$

# APPLICATION OF ANALYSIS

Before the results of the previous section may be applied it is necessary to determine the quantity  $\epsilon$ , the coefficients  $a_N$ , and the reference conditions. The quantities  $\epsilon$  and  $a_N$ , which represent the

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magnitude and form, respectively, of the external velocity distribution, are determined from potential-flow theory or from experimental measurements. Because the results of this report apply primarily to the flow over thin two-dimensional wings at Mach numbers greater than 1,  $\epsilon$  and  $a_{\rm N}$  as obtained by linearized theory (ref. 16) will be presented herein. It is assumed that the coordinates of a wing section are known and can be fitted by a polynomial of fourth or lesser degree:

$$Y = b_1 x + b_2 x^2 + b_3 x^3 + b_4 x^4 \tag{72}$$

The values of  $\epsilon$ ,  $a_N$ , and  $u_r$  are obtained by matching the expression for the velocity distribution obtained from linearized theory with equation (1):

$$\frac{u_e}{u_r} = 1 + \epsilon (a_1 x + a_2 x^2 + a_3 x^3)$$
 (1)

where

$$u_{r} = \left(1 - \frac{a_{1}}{\sqrt{M_{\infty}^{2} - 1}}\right) u_{\infty}$$

$$\varepsilon = \frac{-2b_{2}}{\sqrt{M_{\infty}^{2} - 1}}$$

$$a_{1} = 1 \qquad a_{2} = \frac{3b_{3}}{2b_{2}} \qquad a_{3} = \frac{2b_{4}}{b_{2}}$$
(73)

If the velocity distribution over the wing were known experimentally, the starting point of the calculation would be equation (1), with  $u_r$ ,  $\epsilon$ , and  $a_N$  determined by fitting a polynomial to the measured velocities. If, in a particular application,  $a_2$  or  $a_3$  is much smaller than 1, then that term need not be included in the solution.

The reference Mach number and temperature are obtained from equations (10) and (73):

$$M_{r}^{2} = \frac{M_{\infty}^{2} \left(\frac{u_{r}}{u_{\infty}}\right)^{2}}{1 + \frac{\gamma - 1}{2} M_{\infty}^{2} \left[1 - \left(\frac{u_{r}}{u_{\infty}}\right)^{2}\right]}$$
(74)

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$$t_{r} = t_{\infty} \sqrt{\frac{1 + \frac{\gamma - 1}{2} M_{\infty}^{2}}{1 + \frac{\gamma - 1}{2} M_{r}^{2}}}$$
 (75)

The results of the analysis will now be summarized as they are needed in a particular application. In the following equations the functions  $g_n(\eta)$ ,  $h_n(\eta)$ , and  $H_n(\eta)$  frequently appear. They are related to the tabulated functions in the following manner:

$$g_{N}(\eta) = g_{N1}(\eta) + M_{r}^{2} g_{N2}(\eta) + Kg_{N3}(\eta)$$
 (40)

$$h_N(\eta) = h_{NL}(\eta) + M_r^2 h_{NZ}(\eta)$$
 (44)

$$H_{N}(\eta) = H_{NL}(\eta) + M_{r}^{2} H_{N2}(\eta) + KH_{N3}(\eta) + \frac{K}{M_{r}^{2}} H_{N4}(\eta) + \frac{K^{2}}{M_{r}^{2}} H_{N5}(\eta)$$
(47)

The constant K is related to the given wall temperature for flows with arbitrary rates of heat transfer:

$$K = \frac{1}{s(0)} \left\{ \frac{t_w}{t_r} - 1 - \frac{\gamma - 1}{2} M_r^2 r(0) \right\}$$
 (34)

where s(0) and r(0) appear in table I. (For flows with zero heat transfer, K = 0.)

The velocity profile is given by

$$\frac{\mathbf{u}}{\mathbf{u}_{\mathbf{n}}} = \frac{1}{2} \mathbf{f}'(\eta) + \varepsilon \mathbf{a}_{\mathbf{N}} \mathbf{x}^{\mathbf{N}} \mathbf{g}_{\mathbf{N}}'(\eta)$$
 (51)

where the repeated index N indicates a summation over all values of N. The temperature profile, for zero heat transfer, is

$$\frac{\mathbf{t}}{\mathbf{t_r}} = 1 + \frac{\gamma - 1}{2} \, \mathbf{M_r^2} \, \left[ \mathbf{r}(\eta) - 2\epsilon \, \mathbf{a_N} \mathbf{x^N} \, \mathbf{h_N}(\eta) \right] \tag{52}$$

and for flows with heat transfer, is

$$\frac{t}{t_r} = 1 + Ks(\eta) + \frac{\gamma - 1}{2} M_r^2 \left[ r(\eta) - 2\varepsilon a_{\bar{N}} x^{\bar{N}} H_{\bar{N}}(\eta) \right]$$
 (53)

These profiles can be obtained in terms of the physical variable y by the following relations between  $\eta$  and y: (a) for zero heat transfer,

$$y = 2 \sqrt{\frac{v_r \times C}{u_r}} \left\{ \left[ 1 + \frac{\gamma}{2} M_r^2 \epsilon a_N x^N \right] \left[ \eta + \frac{\gamma - 1}{2} M_r^2 \operatorname{Ir}(\eta) \right] - (\gamma - 1) M_r^2 \epsilon a_N x^N \operatorname{Ih}_N(\eta) \right\}$$
(55)

(b) for arbitrary rates of heat transfer,

$$y = 2 \sqrt{\frac{v_r \times C}{u_r}} \left\{ \left[ 1 + \frac{\gamma}{2} M_r^2 \in a_N^2 \times^N \right] \left[ \eta + \frac{\gamma - 1}{2} M_r^2 \operatorname{Ir}(\eta) + \operatorname{KIs}(\eta) \right] - (\gamma - 1) M_r^2 \in a_N^2 \times^N \operatorname{IH}_N(\eta) \right\}$$
(56)

The functions f, r, and s appear in table II. The functions g appear in table III; h, in table IV; and H, in table V.

The constant C is defined by

$$C = \sqrt{\frac{t_w}{t_r}} \frac{(t_r + 216^{\circ} R)}{(t_w + 216^{\circ} R)}$$
 (3)

where  $t_{\rm w}$  is given for flows with heat transfer, while for zero heat transfer a mean value of the adiabatic wall temperature is used. The adiabatic wall temperature is

$$t_{aw} = t_r \left\{ 1 + \frac{\gamma - 1}{2} M_r^2 \left[ r(0) - 2\varepsilon a_N x^N h_N(0) \right] \right\}$$
 (65)

The temperature recovery factor is

$$F_R = r(0) - 2\epsilon a_N x^N M_r^2 h_{N2}(0)$$
 (67)

The following results were found for local and average skin-friction coefficients and for a heat-transfer parameter:

$$c_{f} = \frac{1}{2} \sqrt{\frac{C}{Re}} \left\{ f''(0) + 2\epsilon a_{N} x^{N} \left[ g_{N}''(0) - \frac{\gamma}{4} M_{r}^{2} f''(0) \right] \right\}$$
 (61)

$$c_{F} = \sqrt{\frac{c}{Re}} \left\{ f''(0) + \frac{2\epsilon e_{N}x^{N}}{2N+1} \left[ g_{N}''(0) - \frac{\gamma}{4} M_{r}^{2} f''(0) \right] \right\}$$
(62)

and

$$Nu = \frac{\sqrt{C \text{ Re}}}{2(t_{aW}^* - t_{W}^*)} \left\{ \text{Ks'(0)} - (\gamma - 1) M_{r}^2 \epsilon a_{N} x^{N} \left[ H_{N}^{i}(0) + \frac{\gamma}{2(\gamma - 1)} \text{ Ks'(0)} \right] \right\}$$
(64)

where

$$Re \equiv \frac{u_r \times v_r}{v_r}$$

$$Nu \equiv \frac{C t_w^* \times v_r}{t_{aw}^* - t_w^*} t_y^* v_w$$

All initial values [f"(0), etc.] are tabulated in table I. The displacement thickness for flows with arbitrary heat-transfer rates is given by

$$\delta^{*} = \sqrt{\frac{\nu_{r} \times C}{u_{r}}} \left\{ \left[ 1 + \frac{\gamma}{2} M_{r}^{2} \epsilon a_{N}^{2} x^{N} \right] \left[ 1.7208 + 4.0218 K + (\gamma-1)(1.1094 M_{r}^{2}) \right] + \epsilon a_{N}^{2} x^{N} \left[ \alpha_{N} + (\gamma-1) M_{r}^{2} B_{N} \right] \right\}$$
(71)

For flows with zero heat transfer, K will vanish and  $B_N$  is replaced by  $\beta_N$  in the last equation. (Values of  $\alpha_N$ ,  $\beta_N$ , and  $B_N$  can be found in table VI.)

The results of this analysis are not necessarily limited to the integral values of N for which calculations were made. Interpolation of the results presented in the tables will yield valid results for other values of N. Values for N=0 are included in order to facilitate this interpolation (see appendix C and table I).

The equations presented in this section apply also for flat-plate flows. For this special case,  $\varepsilon=0$  and the reference conditions are equal to the undisturbed free-stream conditions.

#### DISCUSSION OF EXAMPLES

The results of the previous section were applied to two representative wings in order to determine the combined effects of heat transfer and pressure gradient on boundary-layer characteristics. Cross-sectional views of the forward portion of these wings are shown in figure 4. The

first of the two wings has a constant adverse pressure gradient, while the second has a constant favorable pressure gradient. A maximum thickness ratio of 0.05 and a free-stream Mach number of 3 were chosen for the wing segments of both examples. The velocity and temperature distributions at the outer edge of the boundary layer are shown in figure 5.

The local skin-friction parameter  $c_f\sqrt{Re/C}$  for both representative wings, computed by the present method, is presented in figure 6. The effect of pressure gradient in the absence of heat transfer  $\left(\frac{\partial t}{\partial y}\right)_{tr} = 0$ 

is to decrease skin friction for flows with adverse pressure gradients, and to increase skin friction for flows with favorable gradients. (For flows with zero pressure gradients,  $c_f \sqrt{Re/C} = 0.664$  for all values of x as indicated by a dashed line in the figure.) The effects of pressure gradient are accentuated by adding heat to the boundary layer. For the present examples, the aforementioned decrease and increase in skin friction is doubled when the wall is heated to approximately four times the ambient air temperature. Sufficient cooling at the wall, on the other hand, appears to reverse the trend of the pressure gradient alone. Thus, for a wall temperature approximately equal to one-fourth the ambient air temperature, there is a slight increase in skin friction for flows with adverse pressure gradients; whereas there is a decrease in the case of favorable pressure gradients. The average friction drag parameter  $C_F \sqrt{Re/C}$ , as shown in figure 7, exhibits the same trends as the local skin friction.

The friction-drag parameter  $C_F\sqrt{\text{Re/C}}$  is useful because it applies at all flight altitudes, and the actual velocity, density, and temperature need not be specified a priori. On the other hand, it is a misleading parameter, because the viscosity-temperature dependence factor C, which is a function of the wall temperature ratio, is affected by the rate of heat transfer at the wall. For this reason, the average friction drag coefficient multiplied by  $\sqrt{\text{Re}}$  was found for the two representative wings at conditions existing at 35,000 feet, as shown in figure 8. The rate of change of friction drag along the surface is nearly the same as was shown in figure 7. The relative magnitudes of the friction drag curves are altered, however, so that the highest drag is found for the cooling case, while the lowest drag is obtained with the hot wall, regardless of the type of pressure gradient.

The heat-transfer parameter  $\text{Nu}/\sqrt{\text{Re C}}$  for the two representative wings is plotted in figure 9. The local rate of heat transfer was found to increase along the wing when the pressure gradient and wall temperature were such that the skin friction decreased and vice versa.

The temperature recovery factor, as plotted in figure 10, was found to vary slightly as a result of the pressure gradient. The variation is

of the same order of magnitude as the pressure gradient, and hence a much larger change might be expected for larger pressure gradients. On the other hand, the variable term in the expression for the recovery factor (eq. (67)) is proportional to the square of the Mach number and would be unimportant at low speeds. It is therefore not surprising that recovery factors obtained by the present method do not agree with those obtained in reference 17, where the variation of fluid properties was neglected.

Velocity profiles at the midchord point (x = 1) of the two wings are presented in figure 11. The effect of heat transfer on the local velocity in the boundary layer is seen to be quite large - there is a marked thinning of the boundary layer when heat is extracted and a thickening when heat is added. Although the local velocity and its first derivative are altered only slightly because of the pressure gradient, the local curvature of the profiles appears to be affected to a greater extent. In particular, the profiles for zero heat transfer and a hot wall have an inflection point when the pressure gradient is adverse; whereas no inflection point is evident when the pressure gradient is favorable, even when the wall temperature is four times the ambient air temperature. In general, a velocity profile without an inflection point indicates greater laminar stability than one having an inflection point. (The shape of the temperature profile, however, also affects the criterion of stability.)

Although the local velocity near the outer edge of the boundary layer did not exceed the free-stream velocity, as discussed in reference 18, the functions  $g'(\eta)$  are of a form indicating that such an overshoot may exist for slightly larger pressure gradients. (See fig. 1.)

Temperature profiles for the example wings are plotted in figure 12. These profiles do not differ greatly with the two different pressure gradients. The effect of heat transfer is quite large, however, as is evident from a comparison of the extremely thin profiles when the wall temperature ratio is 0.25 with the relatively thick profiles when this ratio is 4.

The ratio of the displacement thickness along the example wings  $\delta^*$  to the displacement thickness along an equivalent flat plate  $\delta^*_{FP}$  is plotted in figure 13. The displacement thickness is found to be less than the flat plate value for the adverse pressure gradient and greater for the favorable pressure gradient. This behavior is opposite to the trend found for incompressible flow and can be explained as follows: The ratio of displacement thicknesses is found to be

$$\frac{\delta^*}{\delta_{\rm FP}^*} = 1 + \frac{\gamma}{2} \, M_{\rm r}^2 \, \epsilon \, a_{\rm N} x^{\rm N} + \frac{\epsilon \, a_{\rm N} x^{\rm N} \, \left[ \alpha_{\rm N} + (\gamma - 1) \, M_{\rm r}^2 \, B_{\rm N} \right]}{1.72 + 4.02 \, {\rm K} + (\gamma - 1)(1.11) \, M_{\rm r}^2} \tag{76}$$

For incompressible flow and zero heat transfer, equation (76) reduces to

$$\frac{\delta^*}{\delta_{\mathrm{FP}}^*} = 1 - 2.6 \, \varepsilon \, a_{\mathrm{N}} x^{\mathrm{N}} \tag{77}$$

In a favorable gradient ( $\epsilon \, a_N x^N$  positive),  $\delta^*/\delta^*_{FP}$  as expressed by equation (77) decreases; whereas the ratio increases in an adverse gradient. This well-known thinning or thickening of the boundary layer is essentially an effect of the change in local Reynolds number caused by the change in the external velocity.

As the Mach number is increased, however, the term  $\frac{\gamma}{2}\,\mathrm{M}_{r}^{2}\,\,\epsilon\,\,\mathrm{a_{N}}^{N}$  in equation (76) becomes of importance. This term is related to the change in density at the outer edge of the boundary layer. Its significance may be qualitatively determined by supposing for the moment that viscosity may be neglected and by consideration of the two-dimensional compressible vorticity transport equation for an inviscid fluid

$$\frac{D}{Dt} \left( \frac{\Omega}{\rho} \right) = 0 \tag{78}$$

This expression shows that the vorticity changes in the same sense as the density. In a favorable pressure gradient, therefore, the vorticity will decrease along the wing because the density decreases along the wing. A decrease of vorticity in the boundary layer tends to thicken this layer.

If the complete equation for a viscous fluid is considered, there may be two opposing effects which occur at high Mach numbers: The effect of a favorable pressure gradient on Reynolds number (and hence viscosity) tends to thin the boundary layer; at the same time, the effect of the favorable pressure gradient on the vorticity directly tends to thicken the boundary layer. (A similar argument applies to adverse pressure gradients.) At a sufficiently high Mach number this second effect will predominate, as was found in the case of the present examples. For the case of constant pressure gradients and zero heat transfer, it can be shown that the aforementioned reversal of trends in the function  $\delta */\delta^*_{\rm FP}$  occurs at a Mach number of 1.76.

If the Mach number is further increased, the thickening or thinning of the boundary layer will also affect the slope of the velocity profiles at the wall and hence the skin friction. For small constant pressure gradients and zero heat transfer, the skin friction trends are found to reverse at a Mach number of 4.71.

The effect of Prandtl number on the local friction drag parameter over the wing with the adverse pressure gradient is shown in figure 14. At midchord, a Prandtl number of 1 yields a friction drag coefficient 4 percent lower than a Prandtl number of 0.72 when the wall is insulated. A solution for a Prandtl number of 1 for flows with heat transfer was not obtained, but it is expected that the effect would be considerably larger than the 4 percent found for flows with zero heat transfer.

As a check on the accuracy of the present method, the solution for Prandtl number 1 and zero heat transfer was compared with an exact solution of Howarth (refs. 13 and 14), which applies even at pressure gradients as large as required for separation. At the midchord station the local friction drag parameter agrees within 0.7 percent with that obtained by Howarth.

# CONCLUDING REMARKS

A method for the calculation of compressible laminar boundary layer characteristics for flows with heat transfer and small arbitrary pressure gradients is presented. This method was applied to the flow over two representative wings - one with a constant adverse pressure gradient, the other with a constant favorable pressure gradient. The investigation led to the following conclusions: It was found that the deviations in skin friction caused by the pressure gradient were magnified when the wall was heated and reduced when the wall was cooled. Large amounts of cooling were found to reverse the rate of change of skin friction along the wing caused by a pressure gradient alone.

Local rates of heat transfer were found to vary in direct opposition to the skin friction: If the pressure gradient was such that the shearing stress decreased along the wing, then the heat-transfer rate increased, and vice versa.

Temperature recovery factors were found to be affected by the pressure gradient. The percentage change in recovery factor along the wing was somewhat smaller than the percentage change in the external velocity.

The displacement thickness at a Mach number of 3 was found to be greater than the displacement thickness of an equivalent flat plate when the pressure gradient is favorable and less than the flat plate displacement thickness for the adverse pressure gradient. This result is opposite to the trend found at low speeds.

Lewis Flight Propulsion Laboratory
National Advisory Committee for Aeronautics
Cleveland, Ohio, August 19, 1953

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### APPENDIX A

#### SYMBOLS

The following symbols are used in this report:

Prandtl number =  $\mu c_{D}/k$ 

 $A_1, A_2, \ldots$  arbitrary constants (eq. (C2)) a measure of the shape of the external velocity distribution  $\mathbf{a}_{N}$ function appearing in equation (71)  $B_{17}$ b<sub>1</sub>,b<sub>2</sub>,... arbitrary constants (eq. (76)) C constant of proportionality in viscosity-temperature relation average friction drag coefficient =  $\frac{1}{\frac{1}{2}\rho_r u_r^2 x} \int_0^r \tau_w dx$  $C_{\mathbf{F}}$ local friction drag coefficient =  $\frac{\tau_{w}}{\frac{1}{2} \rho_{r} u_{r}^{2}}$  $c_{\mathbf{f}}$ specific heat at constant pressure  $c_p$  $\mathbf{F}_{\mathbf{R}}$ temperature recovery factor f solution of zero-order momentum equation function defined in equations (C4) and (C7) G solution of first-order momentum equation g H solution of first-order energy equation with heat transfer h solution of first-order energy equation without heat transfer factor describing heat-transfer conditions (eq. (34)) K k thermal conductivity M Mach number exponent in free-stream velocity distribution, ( $u_e^* = 1 + \epsilon a_N x^N$ )

Nusselt number =  $\frac{C t_w^* x}{t_{aw}^* - t_w^*} \frac{\partial t^*}{\partial y} \Big)_w$ N Nu transformed variable n

~	P	static pressure
•	q	local rate of heat transfer
	R	gas constant
	Re	Reynolds number = $u_r x/v_r$
2968	r	solution of zero-order energy equation
	s	Sutherland's constant
	s	solution of zero-order energy equation
GR-4 baok	T	total temperature
	t	static temperature
	u	velocity in x-direction
	v	velocity in y-direction
	x	distance along surface measured from leading edge
	Y	normal coordinate of surface
	У	distance from surface measured perpendicular to surface
	$\left. egin{array}{c} lpha_{ m N} \ eta_{ m N} \end{array}  ight\}$	functions appearing in equation (71)
	x	ratio of specific heats
	δ*	displacement thickness
	ε	small quantity - a measure of magnitude of velocity dis- tribution at edge of boundary layer
	η	characteristic variable defined by equation (26)
	θ	dummy variable
	μ	coefficient of viscosity
,	ν	kinematic viscosity = $\mu/\rho$
e	ξ	dummy variable

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ρ mass density

au shearing stress

transformed stream function

w stream function

 $\Omega$  vorticity,  $\left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right)$ 

# Subscripts:

aw adiabatic wall

e conditions at outer edge of boundary layer

FP equivalent flat-plate value

r reference condition

w conditions at wall or surface

∞ undisturbed free-stream condition

x,y,n partial differentiation with respect to x, y, or n

M value of function corresponding to given value of M

N value of function corresponding to given value of N

#### Superscripts:

\* dimensionless quantities defined by equation (11)

differentiation with respect to η

#### Special Notation

A bar over a quantity indicates the order of approximation. (A single bar signifies a zero-order quantity, double bar signifies a first-order quantity, etc.)

A repeated index N appearing on a and one or more other symbols indicates summation:  $1 + \epsilon \ a_N x^N \equiv 1 + \epsilon (a_1 x + a_2 x^2 + \dots)$ .

The symbol I preceding a quantity indicates integration from zero to  $\eta$ : for example,  $Ir(\eta) = \int_{\gamma}^{\eta} r(\xi) d\xi$ .

#### APPENDIX B

# NUMERICAL SOLUTION OF DIFFERENTIAL EQUATIONS

# By Lynn U. Albers

Each of the ordinary differential equations for f, r, s, g, h, and H with its associated boundary conditions at zero and infinity constitutes a two-point boundary value problem. With the exception of the Blasius equation all equations are linear, and the principle of superpositions of any two solutions may be used to satisfy the boundary conditions at infinity. Usually, two solutions close to the correct one were used in the final combination in order to minimize round-off errors. All integrations were performed on the IBM Card-Programmed Electronic Calculator. The combination of solutions and rounding to four decimal places was accomplished on the IBM Type 604 Calculating Punch by using general purpose floating-point control panels.

The integration technique will be described for the g problem, but it will be applicable to all the other problems with slight modifications. If  $g'''(\eta)$  is given at five values of  $\eta$ , a fourth-degree polynomial in  $\eta$  may be passed through the set of values; and if g, g', and g'' are known at the fifth point, the polynomial representation of g''' may be integrated to yield g, g', and g'' at the next (sixth) point. These quantities may then be substituted in the differential equation (41) to yield g'' at the sixth point. Thus, by using the five previous points, the integration may be extended one step at a time.

The integration was initiated with an assumed trial value of g"(0) and a value of g"'(0) calculated from the equation. This value of g"'(0) was also used as a first estimate of g"' at the next four points. The fourth-degree polynomial representing g"' over this range was then integrated to yield g, g', and g" at the second point. Substitution in the equation then yielded a better estimate of g"' at the second point. Integration of the fourth-degree polynomial representation of g"' from zero to successive points was alternated with substitution in the equation to improve values of g"' in an iterative fashion until convergence was obtained at the five initial points.

It was found that when g' was close to its boundary value at infinity, the regular integration process encountered oscillations in the function g''. To avoid this phenomenon, a procedure analogous to the starting procedure was used in an iterative manner. This smoothing process was used from  $\eta=3.4\,$  on. Integration was carried to a point which would yield four-decimal-point accuracy in the value of g'(0) and in the g' and g' data.

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All integrations were performed using a step size of 0.1. Subsequent investigation of the effect of step size indicated that tabular values of the functions f, r, and s are correct as presented in table II, while the functions g, h, and H may be in error by 1 in the fourth place.

#### APPENDIX C

#### SOLUTION OF FIRST ORDER-EQUATIONS FOR N = 0

Physically, the solution of the first-order equations for N=0 is of little interest because the external flow represented by  $u_e^*=1+\epsilon$  is simply the flow over a flat plate; the term  $\epsilon$  arises because the reference velocity is taken slightly different from the stream velocity. In practical applications, flat-plate flows would be handled by the zero-order solutions. The case of N=0 may be of academic interest, however, in addition to supplying limiting conditions for cases of  $N\neq 0$ .

The first-order momentum equation (eq. (38)) for N = 0 becomes

$$g_{Ol}^{""} + f g_{Ol}^{"} + f^{"}g_{Ol} = 0$$
 (C1)  
 $g_{Ol}(0) = g_{Ol}^{i}(0) = 0$   
 $g_{Ol}^{i}(\infty) = 1$ 

The functions  $\mathbf{g}_{02}$  and  $\mathbf{g}_{03}$  vanish identically. The general solution of this equation is

$$g_{O}(\eta) = A_{1}f' + A_{2}(f + f'\eta) + A_{3} \left[ (f + f'\eta) \int_{0}^{\eta} \frac{f'f'' d\eta}{(2f'^{2} - ff'')^{2}} - \int_{0}^{\eta} \frac{(f + f'\eta)f'' d\eta}{(2f'^{2} - ff'')^{2}} \right]$$
(C2)

The coefficient of  $A_2$  in equation (C2) was given in reference 19. From the boundary conditions it can be found that  $A_1 = A_3 = 0$  and  $A_2 = \frac{1}{4}$ . Therefore,

$$g_{01}(\eta) = \frac{1}{4} (f + f'\eta)$$
 (C3)

The first-order energy equation for N=0 and zero heat transfer reduces to

$$h_{Ol}^{"} + Pr f h_{Ol}^{'} = Pr (g_{Ol}r' + f'' g_{Ol}^{"}) = G_{l}(\eta)$$
 (C4)

$$h_{Ol}(0) = 0$$
  $h_{Ol}(\infty) = 1$ 

Equation (C4) is a first-order linear equation in  $h_0^i$ . The solution satisfying the boundary conditions is

$$h_{Ol}(\eta) = 1 - \int_{\eta}^{\infty} \left[f''(\xi)\right]^{Pr} \int_{0}^{\xi} \frac{G_{l}(\theta)}{\left[f''(\theta)\right]^{Pr}} d\theta d\xi \qquad (C5)$$

The function  $h_{O2}$  is identically equal to zero. For flows with arbitrary rates of heat transfer, the following equations arise for N=0:

$$H_{Ol}^{"} + Pr f H_{Ol}^{'} = Pr \left[g_{O}^{r'} + f'' g_{O}^{"}\right] = G_{l}(\eta)$$
 (C6)

$$H_{O4}^{"} + Pr f H_{O4}^{!} = \frac{2 Pr}{\gamma - 1} g_{O} s^{!} = G_{4}(\eta)$$
 (C7)

$$H_0(0) = 0$$
  $H_{01}(\infty) = 1$   $H_{04}(\infty) = 0$ 

$$H_{O2} = H_{O3} = H_{O5} \equiv 0$$

The solution of equation (C6) satisfying the appropriate boundary conditions is

$$H_{\text{Ol}}(\eta) = 1 - \int_{\eta}^{\infty} [f''(\xi)]^{\text{Pr}} \int_{0}^{\xi} [f''(\theta)]^{-\text{Pr}} G_{l}(\theta) d\theta d\xi -$$

$$\frac{1 - \int_{0}^{\infty} [f''(\xi)]^{Pr} \int_{0}^{\xi} [f''(\theta)]^{-Pr} G_{1}(\theta) d\theta d\xi}{\int_{0}^{\infty} [f''(\xi)]^{Pr} d\xi} \int_{\eta}^{\infty} [f''(\xi)]^{Pr} d\xi$$
(C8)

Similarly, the solution of equation (C7) is

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$$H_{04}(\eta) = - \int_{\eta}^{\infty} [f''(\xi)]^{Pr} \int_{0}^{\xi} [f''(\theta)]^{-Pr} G_{4}(\theta) d\theta d\xi +$$

$$\frac{\int_{0}^{\infty} \left[f''(\xi)\right]^{\operatorname{Pr}} \int_{0}^{\xi} \left[f''(\theta)\right]^{-\operatorname{Pr}} G_{4}(\theta) \ d\theta \ d\xi}{\int_{0}^{\infty} \left[f''(\xi)\right]^{\operatorname{Pr}} d\xi} \int_{\eta}^{\infty} \left[f''(\xi)\right]^{\operatorname{Pr}} d\xi \tag{C9}$$

The function  $H_O(\eta)$  is again obtained by a linear combination of  $H_{O1}(\eta)$  and  $H_{O4}(\eta)$  in the following manner:

$$H_{O}(\eta) = H_{Ol}(\eta) + \frac{K}{M_{r}^{2}} H_{O4}(\eta)$$
 (C10)

Values of  $h_0(0)$  and  $H_0'(0)$  for Pr = 0.72 were obtained by numerical integration and are listed in table I.

#### APPENDIX D

#### SOLUTION FOR Pr = 1

In order to establish the effect of Prandtl number on skin friction and to provide a basis for comparison with other solutions, some of the energy equations were solved for the special case of Pr=1. The solution of the zero-order energy equation for Pr=1 is

 $r(\eta) = 1 - \frac{1}{4} (f')^2$  (D1)

$$s(\eta) = 2 - f'(\eta) \tag{D2}$$

The solution of the first-order energy equations for zero heat transfer is, for Pr = 1,

 $h_{NL}(\eta) = \frac{1}{2} f' g_{NL}^{\dagger}$   $h_{N2}(\eta) = \frac{1}{2} f' g_{N2}^{\dagger}$ (D3)

and

The linear combination of equations (D3) yields

$$h_{N}(\eta) = \frac{1}{2} f' \left[ g_{NL}^{i} + M_{T}^{2} g_{N2}^{i} \right]$$
 (D4)

The function  $\mathbf{g}_{\mathrm{Nl}}$  is independent of Prandtl number, and hence the values appearing in table III apply for all Prandtl numbers. The function  $\mathbf{g}_{\mathrm{l2}}$  was calculated numerically for a Prandtl number of 1, and results of this calculation appear in table VII.

The complete solution of the first-order energy equation for Pr=1 and flows with arbitrary rates of heat transfer was not found. The following are the solutions of equations (45) and (46) for Pr=1 and heat transfer:

$$H_{N\perp}(\eta) = \frac{1}{2} f' g_{N\perp}^{t}$$

$$H_{N2}(\eta) = \frac{1}{2} f' g_{N2}^{t}$$
(D5)

and for Pr = 1 and N = 0:

$$H_{O4}(\eta) = \frac{\eta f''(\eta)}{2(\gamma - 1)}$$
 (D6)

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## TABLE I. - INITIAL VALUES

[Pr, 0.72; γ, 1.4]

f''(0) = 1.3282

r(0) = 0.8477

s(0) = 2.0748

s'(0) = -1.2267

	N = 0	N = 1	N = 2	N = 3
g"(0)	0.9962	4.0821	6.3546	8.2879
g"(0)	0	.2807	.5847	.8717
g"(0)	0	5.0447	8.9738	12.4065
h <sub>N1</sub> (0)	0.1523	0.1524	0.1526	0.1528
h <sub>N2</sub> (0)	0	.0085	.0123	.0146
H'N1(0)	0.0904	0.1479	0.1802	0.2042
H <sup>MS</sup> (0)	o	.0082	.0145	.0195
H <sub>1</sub> (0)	o	4201	2200	.0899
H <sub>N4</sub> (0)	1.5326	5.5574	8.6985	11.3879
H; (0)	0	5.3452	10.1820	14.5535

TABLE II. - SOLUTIONS OF ZERO-ORDER MOMENTUM AND EMERGY EQUATIONS

[Pr, 0.72]

					, 0.72]		~~~~~		
η	ŕ	Į,	t.	Ir	r	r i	Īβ	В	<b>5</b> †
ں 1 22 3 4	0,0000 .0066 .0866 .0597 .1061	0.0000 1326 2655 3979 5394	1.3282 1.3279 1.3269 1.3205 1.3096	0.0000 .0047 .1667 .2514 .3323	0.8477 .8445 .8350 .8194 .797%	0.0000 0635 1868 1893 3503	0.0000 .2013 .3904 .5672 .7319	2.0740 1,9531 1,8395 1,7071 1,5853	-1.3 3 6 7 -1.3 3 6 5 -1.3 3 5 2 -1.3 3 1 5 -1.3 1 4 3
5 6 7 8 8	1699 3370 32803 48895	6596 7875 9123 10335 11495	1,8980 1,8663 1,8314 1,1866 1,1317	.4107 .4860 .5576 .6252 .6863	.7692 .7366 .6969 .6536	3066 3686 4110 4588 4846	.8843 1,0346 1,1534 1,8704 1,3761	1,4644 1,3450 1,8276 1,1189 1.0016	-12020 -1.1653 -1.1617 -1.1311 -10933
1.0 1.1 1.3 1.4	.6500 .7012 .9823 1.0725 1.8310	1.2595 1.3626 1.4579 1.5449 1.6 230	1,0670 ,99,24 ,9184 ,0259 ,7361	.7404 .7996 .6476 .by03 .ys60	.5570 .5057 .4536 .4030 .3519	5071 6189 5198 6101 4906	1.4709 1.5552 1.6295 1.0945 1.7508	.8945 .7983 .6957 .6058 .5215	-10476 -9958 -9361 -8713 -8020
1.6 1.7 1.8 1.5	1.3968 1.5691 1.7469 1.9295 21160	1,6921 1,7522 1,6035 1,9467 1,8823	.6455 .5566 .4715 .7954 .3205	9608 .9690 1.0120 1.0326 1.0494	.3648 .8596 .8187 .1019	4627 4288 3889 3470 3048	1,7990 1,8400 1,8744 1,9030 1,9265	.4449 .3757 .3138 .8592 .8117	-7296 -550 -5820 -5099 -4408
2.0 2.1 2.3 2.4 2.4	8.3057 8.4930 2.6524 2.8683 5.0853	19110 19339 19517 19684 19756	2569 2081 1509 1179 10075	1.0689 1.0737 1.0884 1.0891 1.0944	.1211 .0969 .0765 .0597	2623 2223 1854 1522 1231	19456 19609 19730 19826 19899	.1709 .1364 .1075 .0837	3759 3168 3685 8146 1730
995% 993% 993 993	3.8 & 3.3 3.4 & 1.9 3.6 & 6.9 3.8 & 6.3 4.0 7 9 9	1,9031 1,9085 1,9983 1,9960 1,9967	.0556 .0454 .0317 .0217	1.09 6 4 1.1 0 1 4 2.1 0 3 7 1.1 0 5 4 1.1 0 6 6	.0349 .036% .0194 .0143	09 61 07 70 05 9 5 04 64 03 41	1,9956 1,9998 2,0030 2,0053 2,0070	.0489 .0366 .0271 .0198	1376 1078 0833 0635 0477
3.0 3.1 3.2 3.3 3.4	43796 44794 46793 48793 50793	1,9975 1,9967 1,5593 1,5995 1,5997	.00% 6 .00% 6 .0039 .00% 8 .0018	1.1075 1.1061 1.1065 1.1064 1.1090	.0077 .0051 .0035 .0027	0252 0164 0152 0094 0065	20083 20091 30097 30101 20104	.0102 .0072 .0050 .0034 .0023	0355 0357 0185 0131 0093
3.6 3.6 3.7 3.8 3.9 5.5	5.8 79 8 5.4 79 8 5.6 79 8 5.5 79 8 6.0 79 3	1,9590 1,9599 2,0000 2,0000 2,0000	.0009 .0005 .0005 .0007	11 09 1 11 09 2 11 09 3 11 09 3 11 09 3	.0011 .0007 .0004 .0002	0045 0031 0050 0013 0009	20100 20107 20108 20108 20108	.0015 .0010 .0006 .0004	-2065 -2045 -2026 -2010 -2018
4.0 4.1 4.3 4.4	6.3 7 9 3 6.4 7 9 2 6.6 7 9 3 6.3 7 9 2 7.0 7 9 2	2,0000 2,0000 2,0000 2,0000 2,0000	.0000 .0000 .0000 .0000	11 09 3 11 09 3 11 09 3 11 09 3 11 09 4	.0001 .0001 .0000 .0000	0003 0003 0008 0001	8.0109 8.0109 8.0109 8.0109 8.0109	.0002 .0001 .0001 .0000	-,0000 -,0005 -,0003 -,0003 -,0001
4.m	75.798	a.0000	.0000	1.1094	.0000	0001	0109	.0000	001

TABLE III. - SOLUTIONS OF FIRST-ORDER NOMENTUM EQUATION

		[Pr, 0.72; 7, 1.40]							NAGA	•
η	s <sub>ll</sub>	<b>5</b> 12	813	811	<b>5</b> 12	813	ซ <sub>ี่ไไ</sub>	812	B13	N
0 .1 .8 .3 .4	0.0000 .0197 .0763 .1657 .2839	0.0000 .0013 .0047 .0096 .0154	0,0000 ,0839 ,0902 ,1913 3803	0.0000 3882 .7364 1.0446 1.3189	0.0000 .0847 .0487 .0543 .0599	0.0000 .4638 .8495 11680 1,4063	4.0 8 2 1 3.6 8 2 1 3.2 8 2 1 2.8 8 2 3 2.4 8 2 9	0.2807 .2133 .1474 .0851 .0379	5.0 4 4 7 4.2 3 9 4 3.4 8 3 1 2.7 7 5 8 2.1 1 7 6	1 1 1 1 1
.5 .6 .7 .8	.4869 .5908 .7716 .9655 1.1685	.0 8 1 4 .0 8 7 8 .0 3 8 4 .0 3 6 6 .0 3 9 7	.4705 .6358 .8106 .9900 1.1693	1.5 41 3 1.7 3 0 0 1.8 7 9 5 1.9 9 0 7 3.0 6 4 8	.0601 .0557 .0475 .0365 .0237	1.5872 17098 17790 18003 17791	20849 16903 13018 9238 .5619	0225 0647 0977 1207 1334	1,5089 .9505 .4438 0089 4058	1 1 1 1
1.0 1.1 1.3 1.3 1.4	1.3773 1.5882 1.7984 8.0050 2.3057	.0413 .0417 .0407 .0386 .0356	13446 15125 16703 18159 19478	31038 31104 3.0878 8.0403 1.9784	.0101 0032 0155 0261 0344	1.7313 1.6388 1.5800 1.3893 1.2470	.2 2 2 8 0 8 6 2 3 5 7 7 5 8 5 1 7 6 3 5	1360 1395 1152 0951 0712	7428 -1.0171 -1.2382 -1.3752 -1.4597	111111
1.5 1.6 1.7 1.8 1.9	2.3989 2.5833 2.7580 2.9228 3.0779	.0318 .0376 .0332 .0188 .0146	2,0651 2,1677 2,3556 2,3297 2,3911	1.8893 1.7961 1.6978 1.5990 1.5036	0403 0436 0446 0435 0408	1,0993 .9516 .8089 .6751 .5531	8 90 4 9658 9988 9770 9858	0458 0811 .0018 .0198 .0338	-1.4856 -1.4591 -1.3884 -1.8838 -1.1540	1 1 1 1 1
2.0 2.1 2.3 2.3 2.4	3.2237 3.3611 3.4910 3.6143 3.7382	.0107 .0072 .0042 .0017 0003	2,4408 2,4805 2,5115 2,5353 2,5533	1.4147 1.3345 18643 12046 11551	0369 0383 0874 0886 0108	.4448 .3510 .2719 .2067 .1543	8 4 8 8 7 5 3 3 6 4 9 7 5 4 5 0 4 4 5 1	.0 4 3 1 .0 4 7 9 .0 4 8 8 .0 4 6 6 .0 4 2 4	-1.0110 8637 7200 5861 4661	111111
2.5 2.6 2.7 2.8 2.9	3,8 4 5 7 3,9 8 5 6 4,0 6 2 7 4,1 6 7 7 4,2 7 1 8	0019 0038 0041 0048 0053	2.5666 2.5768 2.5830 2.5878 2.5911	11153 1.0339 1,0598 1,0418 1,0286	0148 0108 0080 0058 0041	1189 0811 0578 0397 0870	-3543 -8750 -2083 -1541 -1113	.0368 .0308 .0349 .0195	-3624 -2755 -2050 -1493 -1064	1 1 1 1 1
3.0 3.1 3.2 3.3 3.4	4.3736 4.4751 4.5768 4.6768 4.7772	0056 0059 0060 0061 0063	25948 25958 25964 25968	1.0192 1.0126 1.0081 1.0081 1.0032	0028 0019 0013 0008 0005	0180 0118 0076 0048 0030	-0786 -0548 -0366 -0341 -0155	.0109 .0078 .0054 .0037	0744 0509 0342 0324 0144	11111
3.5 3.6 3.7 3.8 3.9	4,8775 4,9776 5,0777 5,1778 5,2778	0068 0063 0063 0063	25970 25971 35978 25973 25973	1.0019 1.0011 1.0007 1.0004 1.0008	0003 0003 0001 0001	0018 0011 0007 0004 0003	0098 0061 0037 0038 0013	.0016 .0010 .0006 .0003	0093 0057 0035 0021 0018	1 1 1 1
4.0 4.1 4.8 4.3 4.4	5.3778 5.4778 5.5778 5.6778 5.7778	-0063 -0063 -0063 -0063	25973 25973 25973 25973 25973	1.0001 1.0001 1.0000 1.0000	.0000	.0001 .0001 .0000 .0000	0007 0004 0008 0001 0001	.0001 .0001 .0000 .0000	0007 0004 0003 0001	1 1 1 1 1
4.5	5.8778	0063	25973	1.0000	.0000	.0000.	.0000	.0000	0000	1

TABLE III. - Continued. SOLUTIONS OF FIRST-ORDER MOMENTUM EQUATION

[Pr. 0.72: v. 1.40]

					(Pr. 0.72; η	, 1.40]			~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~	
η	g <sub>21</sub>	522	6 <sub>23</sub>	8 <sub>21</sub>	8 <sub>22</sub>	8 <sub>25</sub>	8 <sub>21</sub>	8 <mark>22</mark>	g" 23	N
0 .1 .8 .3	0.0000 .0304 .1165 .2603 .4248	0.0000 .0027 .0099 .0803 .0327	0,0000 .0421 .1581 .3328 .5529	0.0000 .5965 11119 15512 19164	0.0000 .0517 .0901 .1159	0.0000 .8161 1.4772 1.9957 8.3848	6,3 5 4 6 5,5 8 7 3 4.7 7 4 3 4.0 1 6 7 3.2 9 1 7	0.5847 4499 3197 1981 .0884	89738 73667 58766 43152 38878	& W W W W
.5 .6 .7 .8	.6312 .8642 1.1168 1.3830 1.6572	.0460 .0592 .0716 .0826	.8059 1.0810 1.3685 1.6601 1.9485	2.2108 2.4385 2.6037 2.7108 2.7647	.1341 .1893 .1175 .1005 .0801	2.6577 2.8880 2.9083 2.9109 2.8477	2,6041 1,9573 1,3538 .7964 .2886	0069 0858 1471 1903 2157	8.1948 1,2316 ,3947 -,3819 -,9833	8888
1.0 1.1 1.2 1.3 1.4	1,9343 2,2098 2,4798 2,7409 2,9906	.0985 .1032 .1057 .1062 .1050	22 278 24930 27404 29670 31711	2.7703 2.7335 2.6604 2.5575 2.4319	.0579 .0357 .0147 0039 0193	8.7299 8.5685 8.3738 8.1560 1.9247	-1654 -5604 -8914 -11542 -13461	2343 2181 1994 1712 1370	-1.4140 -1.7979 -2.0786 -2.3607 -2.3501	22 22 23 23
15 16 17 18 19	3.8268 3.4484 3.6549 3.8463 4.0832	.1025 .0989 .0947 .0908 .0857	3,3 5 1 8 3,5 0 9 0 3,6 4 3 4 3,7 5 6 3 3,8 4 9 4	22907 21408 19887 18403 1.7002	0312 0393 0439 0454 0443	1.6888 1.4562 1.2337 1.0267 8392	-1.4670 -1.5200 -1.5113 -1.4498 -1.3465	0999 0633 0296 0008 .0218	-2.3549 -8.3857 -8.1553 -1.9780 -1.7690	20 20 20 20 20
9.0 9.1 2.2 2.3 2.4	41867 43381 44788 46105 47347	.0814 .0775 .0741 .0711	3,9 2 4 8 3,9 8 4 9 4,0 3 1 7 4,0 6 7 7 4,0 9 4 7	1.5780 1.4581 1.3596 1.8767 1.8086	0413 0369 0319 0267 0216	.6735 .5307 .4105 .3117 .2383	-1,8136 -1,0638 -9065 -,7589 -6097	.0379 .0477 .0520 .0518 .0484	-1.5431 -1.3137 -1.0988 8878 7044	00000000000000000000000000000000000000
2.5 2.6 2.7 2.8 2.9	48587 49659 50754 51880 52866	.0668 .0653 .0648 .0633	4.1147 4.1292 4.1395 4.1467 4.1517	11542 11117 10793 10552 10376	0171 0131 0098 0071 0051	1700 1881 .0861 .0596	-4816 -3713 -2796 -2057 -1478	.0489 .0365 .0298 .0255	5468 4153 3087 2346 1600	3333
3.0 3.1 3.8 3.3 3.4	5,3897 5,4918 5,5931 5,6940 5,7945	.0623 .0620 .0618 .0617	4.1550 4.1573 4.1587 4.1596 4.1608	1.0252 1.0165 1.0106 1.0066 1.0041	0035 0034 0016 0010 0006	.0270 .0177 .0114 .0072 .0045	-1039 -0714 -0481 -0314 -0303	.0133 .0096 .0067 .0045	-1118 -0764 -0514 -0336 -0217	8888
3.5 3.6 3.7 3.8 3.9	5.8948 59950 60951 61952 62982	.0616 .0615 .0615 .0615	4.1605 4.1607 4.1609 4.1609 4.1610	1.0025 1.0015 1.0006 1.0005 1.0003	0004 0008 0001 0001	0027 0016 0010 0006 0003	-D127 -0079 -D047 -D028 -D016	.0019 .0012 .0007 .0004 .0002	0138 0086 0058 0038 0018	*****
4.0 4.1 4.3 4.3 4.4	6,3953 6,4983 6,5953 6,6953 6,7983	.0615 .0615 .0615 .0615	4.1610 4.1610 4.1610 4.1610 4.1610	1.0001 1.0001 1.0000 1.0000 1.0000	.0000 .0000 .0000 .0000	0008 0001 .0000 0000	0009 0005 0003 0001 0001	.0001 .0001 .0000 .0000	0011 0006 0004 0003 0001	88888
4.5	6.8953	-0615	4.1610	1,0000	.0000	0000	0000	0000	.0000	3

TABLE III: - Concluded. SOLUTIONS OF FIRST-ORDER MOMENTUM EQUATION

	[Pr, 0.72; $\gamma$ , 1.40]								NACA	•
n	g <sub>31</sub>	832	g <sub>33</sub>	851	B <sup>1</sup> <sub>32</sub>	5 <sub>35</sub>	851	832	8 <sub>33</sub>	н
0 1954	0.0000 .0394 .1499 .3197 .5381	0.0000 0040 0147 0303 0488	0.0000 .0579 .2161 .4523 .7469	0.0000 .7690 1.4203 1.9587 8.3916	0.0000 .0770 1343 .1789 .1947	0.0000 1.1189 2.0067 2.6850 31772	8,2 8 7 9 7,0 9 4 7 5,9 3 8 2 4,8 4 4 8 3,8 2 8 8	0.8717 .6698 .4766 .2986 .1409	13.4065 10.0003 7.7924 5.8129 4.0736	35533
.5 .6 .7 .8 .9	.7948 1.0806 1.3869 1.7063 2.0319	.0687 .0887 .1077 .1249 .1396	1.0823 1.4437 1.8181 2.1947 2.5646	2.7278 2.9739 3.1405 3.2353 3.8665	8018 1968 1881 1603 1339	3.5073 3.6982 3.7713 3.7460 3.6399	28970 20527 12933 .6163 .0192	.0064 1089 1865 2449 2794	2.5677 1.8856 .2079 6847 -1.4106	กรรร
1.0 1.1 1.9 1.3	8.3577 8.6786 8.9902 3.8890 3.5721	.1516 .1607 .1669 .1704 .1717	2.9805 3.2567 3.5687 3.8536 4.1093	3.2 418 3.1693 3.0571 3.9138 2.7460	.1052 .0761 .0485 .0236	3.4 68 9 3.2 4 7 4 2.9 8 8 7 8.7 0 5 2 2.4 0 8 3	4988 9373 -1.2945 -1.5689 -1.7600	2922 2861 2645 2313 1906	-1.9855 -3.4230 -3.7306 -3.9203 -3.0003	33333
1.5 1.6 1.7 1.8 1.9	3.8377 4.0847 4.3186 4.5880 4.7137	.1711 .1690 .1659 .1681 .1581	4,3 3 5 1 4,5 3 1 8 4,6 9 8 6 4,8 3 9 0 4,9 5 4 7	2.5 6 3 9 2.3 7 4 7 2.1 8 5 8 2.0 0 3 5 1.8 3 3 0	0144 0268 0349 0392 0408	8.1084 1.8149 1.5365 1.2765 1.0425	-1.8696 -1.9088 -1.8658 -1.7718 -1.6315	1463 1021 0610 0254 .0033	-3.9 6 1 5 -2.8 7 6 7 -2.7 0 0 9 -2.4 7 1 1 -3.8 0 5 1	3 3 3 3
2.0 2.1 2.3 2.3 2.4	4.8892 5.0500 5.1981 5.3354 5.4638	.1548 .1504 .1471 .1442 .1418	5.0484 5.1289 5.1810 5.2256 5.2591	1.6782 1.5415 1.4239 1.3253 1.2448	0388 0356 0313 0265 0818	.8361 .6585 .5091 .3864 .8879	-1.4606 -1.2725 -1.0798 8933 7208	.0245 .0385 .0460 .0483 .0466	-1.9 2 0 4 -1.6 3 3 1 -1.3 5 6 6 -1.1 0 1 3 8 7 3 9	3 3 3 3 3
2.5 2.6 2.7 2.8 2.9	5.5849 5.7004 5.8114 5.9198 6.0245	.1398 .1383 .1371 .1363 .1356	5.2 8 3 9 5.3 0 1 9 5.3 1 4 6 5.3 8 3 6 5.3 2 9 7	11805 11305 10985 10648 10437	0173 0134 0100 0073 0058	2106 1512 1066 .0738 0501	5676 4364 3877 3406 1735	.0428 .0365 .0301 .0841	6782 5149 3825 3783 1983	3 3 3 3 3
3.0 3.1 3.8 3.3 3.4	6.1281 6.2305 6.3320 6.4330 6.5336	.1352 .1349 .1347 .1346 .1345	5,3338 5,3366 5,3383 5,3395 5,3408	1,0292 1,0191, 1,0122 1,0076 1,0047	0036 0034 0016 0010	.0334 .0219 .0141 .0089 .0055	1211 0830 0560 0363 0833	.0139 .0098 .0073 .0043	1385 0946 0638 0413 0268	3 3 3 3 3
3.5 3.6 3.7 3.8 3.9	6.6340 6.7348 68343 6.9344 7.0345	.1344 .1344 .1344 .1344	53406 53409 53410 53411 53418	1,0038 1,0017 1,0010 1,0005	0004 0002 0001 0001	.0034 .0020 .0012 .0007 .0004	0147 0091 0055 0033 0019	.0020 .0012 .0007 .0004 .0003	0170 0106 0064 0040 0023	3 3 3 3 3
4-0 4-1 4-8 4-3 4-4	7.1.345 7.2345 7.3345 7.4346 7.5345	.1344 .1344 .1344 .1344	53418 53418 53418 53418 53418	1,0001 1,0001 1.0000 1.0000	.0000 .0000 .0000 .0000	0002 0001 0000 0000	0011 0005 0003 0001	.0001 .0001 .0001 .0000	0014 0007 0005 0003 0008	3 3 3 3
4.5	7-6345	.1344	53412	1.0000	0000	0000	.0000	.0000	٥٥٥٥	3

TABLE IV. - SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR ZERO HEAT TRANSFER

[Pr, 0.72; γ, 1.40]						NACA	
η	Ih <sub>11</sub>	<sup>Ih</sup> 12	h <sub>11</sub>	h <sub>12</sub>	h <sub>i1</sub>	h <sub>12</sub>	N
0 1 2 3 4	0.0 0 0 0 .0 1 5 9 .0 3 5 3 .0 6 1 4 .0 9 6 8	0.0000 .0009 .0020 .0034 .0053	0.1524 .1710 .2232 .3034 .4059	0.0085 .0097 .0126 .0163 .0200	0.0000 .3631 .6712 .9231 11174	0.0000 .0221 .0348 .0386	1 11 11 11 11 11 11 11 11 11 11 11 11 1
.5 .6 .7 .8 .9	1432 2021 2742 3598 4583	.0074 .0098 .0123 .0146	.5249 .6544 .7886 .9215 1.0479	.0230 .0246 .0244 .0223 .0183	12526 13282 13448 13051 12139	.0236 .0078 0111 0309 0494	11111
1.0 1.1 1.2 1.3 1.4	.5690 .6904 .8208 .9584 1.1012	.0182 .0191 .0193 .0187	1.1628 1.2624 1.3437 1.4051 1.4460	.0126 .0055 0024 0104 0182	1,0784 .9083 .7151 .5118 .3091	0648 0755 0806 0799 0737	1 1 1 1 1 1
1.5 1.6 1.7 1.8 1.9	1.2470 1.3940 1.5406 1.6854 1.8273	.0151 .0123 .0090 .0054 .0016	1.4673 1.4708 1.4589 1.4347 1.4015	0250 0306 0347 0372 0382	.1200 0470 1856 2925 3671	0629 0489 0331 0171 0021	111111
20 21 22 23 24	1.9655 2.0996 2.2295 2.3551 2.4767	0022 0059 0094 0126 0155	1.3623 1.3201 1.2772 1.2356 1.1967	0377 0361 0335 0304 0269	4114 4291 4251 4044 3720	.0109 .0214 .0290 .0338 .0361	111111
2.5 2.6 2.7 2.8 2.9	2.5946 2.7091 2.8208 2.9299 3.0371	0180 0201 0219 0234 0246	1.1614 1.1303 1.1035 1.0809 1.0622	0232 0197 0163 0133 0106	3326 2898 2467 2056 1679	.0363 .0348 .0321 .0286 .0248	1 1 1 1
3.0 3.1 3.2 3.3 3.4	3.1 4 2 5 3.2 4 6 6 3.3 4 9 6 3.4 5 1 9 3.5 5 3 4	0255 0263 0268 0272 0276	1.0 4 7 2 1.0 3 5 2 1.0 2 5 9 1.0 1 8 7 1.0 1 3 4	0083 0064 0049 0036 0026	1345 1057 0816 0619 0464	.0210 .0173 .0139 .0110 .0085	1 1 1 1
3.5 3.6 3.7 3.8 3.9	3.6 5 4 6 3.7 5 5 4 3.8 5 5 9 3.9 5 6 3 4.0 5 6 5	0278 0279 0281 0281 0282	1.0094 1.0065 1.0044 1.0030 1.0030	0019 0013 0009 0006 0004	0338 0244 0173 0120 0082	.0064 .0048 .0035 .0035	1 1 1 1
4.0 4.1 4.2 4.3 4.4	4.1 5 6 7 4.2 5 6 8 4.3 5 6 8 4.4 5 6 9 4.5 5 6 9	0282 0282 0283 0283	1.0013 1.0008 1.0005 1.0003 1.0002	0003 0002 0001 0001	0056 0037 0024 0016 0010	.0018 .0008 .0006 .0004 .0002	1 1 1 1
4.5 4.6 4.7 4.8 4.9	4.6569 4.7569 4.8570 4.9570 5.0570	0283 0283 0283 0283	1.0001 1.0001 1.0001 1.0000 1.0000	.0000 .0000 .0000 .0000	0006 0004 0002 0001 .0000	.0002 .0001 .0001 .0000	1 1 1 1

TABLE IV. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR ZERO HEAT TRANSFER

			ACA				
η	Ih <sub>21</sub>	Ih <sub>22</sub>	h <sub>21</sub>	h <sub>22</sub>	h <sub>2</sub> 1	h¿2	И
0 123 4	0.0 0 0 0 0.1 6 2 0.3 7 9 0.6 9 5 1.1 4 3	0.0000 .0013 .0031 .0056 .0089	0.1526 1812 2597 .3774 .5239	0.0123 .0148 .0210 .0291 .0375	0.0000 .5533 .9986 13381 1.5753	0.0000 .0465 .0744 .0852 .0808	3223
.5 .6 .7 .8	1749 2525 3477 .4600 .5885	.0130 .0178 .0230 .0282 .0331	.6892 .8638 1.0390 1.2067 1.3600	.0449 .0500 .0522 .0512 .0468	1,7149 1,7628 1,7267 1,6159 1,4419	.0639 .0379 .0061 0275 0595	2 2 2 2 2
1.0 1.1 1.2 1.3 1.4	.7314 .8864 10510 12224 13979	.0 3 7 4 .0 4 0 9 .0 4 3 3 .0 4 4 5 .0 4 4 6	1.4934 1.6025 1.6847 1.7390 1.7660	.0394 .0297 .0183 .0063	1.2181 9597 .6830 .4044 .1392	0867 -1069 -1185 -1212 -1153	2222
1.5 1.6 1.7 1.8 1.9	1.5747 1.7507 1.9236 2.0920 2.2547	.0434 .0413 .0383 .0347 .0307	1.7 6 7 7 1.7 4 7 4 1.7 0 8 9 1.6 5 6 8 1.5 9 5 3	0165 0359 0332 0384 0414	0993 3013 4607 5753 6464	1023 0840 0627 0405 0191	2 2 2 2
a.0 a.1 a.3 a.4	2.4109 2.5604 2.7031 2.8394 2.9696	.0265 .0223 .0183 .0145	1.5 2 8 8 1.4 6 0 8 1.3 9 4 3 1.3 3 1 7 1.2 7 4 3	0423 0415 0393 0362 0324	6782 6767 6490 6022 5432	0002 .0154 .0273 .0354 .0400	322
2.5 2.6 2.7 2.8 2.9	3.0 9 4 4 3.2 1 4 5 3.3 3 0 4 3.4 4 2 9 3.5 5 2 5	.0080 .0054 .0032 .0014	1.2232 1.1788 1.1410 1.1095 1.0838	0283 0241 0302 0165 0133	4778 4107 3456 2851 2307	.0416 .0408 .0383 .0347 .0304	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
3.0 3.1 3.2 3.3 3.4	3.6598 3.7653 3.8693 3.9782 4.0743	0013 0022 0029 0035 0039	1.0631 1.0469 1.0343 1.0247 1.0176	0104 0081 0061 0046 0034	-1833 -1431 -1097 -0828 -0614	.0259 .0215 .0174 .0138 .0107	2 2 2 2 2
3.5 3.6 3.7 3.8 3.9	4.1 7 5 8 4.2 7 6 8 4.3 7 7 6 4.4 7 8 0 4.5 7 8 3	0042 0044 0045 0046 0047	1.0123 1.0085 1.0058 1.0039 1.0026	0024 0017 0012 0008 0005	0448 0321 0227 0157 0107	.0081 .0061 .0044 .0032	2 2 2 2 2 2
4.0 4.1 4.2 4.3 4.4	4.6786 4.7787 4.8788 4.9788 5.0789	0047 0048 0048 0048	1.0017 1.0011 1.0007 1.0004 1.0003	0004 0002 0001 0001	0072 0048 0031 0020 0014	.0016 .0011 .0007 .0005 .0003	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
4.5 4.6 4.7 4.8 4.9	5.1 789 5.2 789 5.3 789 5.4 789 5.5 789	0048 0048 0048 0048 0048	1.0002 1.0001 1.0001 1.0001 1.0000	.0000 .0000 .0000 .0000	0008 0005 0003 0002 0001	.0002 .0002 .0001 .0001	2 2 2 2 2

TABLE IV. - Concluded. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR ZERO HEAT TRANSFER

[Pr, 0.72; γ, 1.40]								
η	Ih <sub>31</sub>	Ih <sub>32</sub>	h <sub>31</sub>	h <sub>32</sub>	h51	h32	N	
0 1 2 3 .4	0.0000 .0165 .0400 .0761 .1286	0.0000 .0016 .0038 .0072 .0118	0.1528 .1897 .2897 .4371 .6172	0.0146 .0183 .0275 .0396 .0524	0.0000 .7112 12623 16612 19186	0.0000 .0693 .1111 .1279 .1231	3 3 3 3	
.5 .6 .7 .8	.2001 .2921 .4046 .5368 .6870	.0176 .0244 .0319 .0396 .0471	.8165 1.0229 1.2255 1.4154 1.5848	.0637 .0721 .0767 .0769 .0727	2.0473 2.0616 1.9764 1.8073 1.5710	.1009 .0663 .0242 0202 0624	3 3 3 3 3	
1.0 1.1 1.2 1.3 1.4	.8528 1.0315 1.2199 1.4146 1.6125	.0540 .0599 .0646 .0678	1.7279 1.8407 1.9209 1.9683 1.9841	.0646 .0534 .0398 .0251	1.2849 0.9674 0.6371 0.3121 0.0090	0987 1261 1429 1486 1440	3 3 3 3 3	
1.5 1.6 1.7 1.8 1.9	1.8105 2.0059 2.1966 2.3809 2.5574	.0700 .0691 .0670 .0641 .0606	1.9713 1.9339 1.8770 1.8056 1.7251	0034 0154 0253 0327 0375	2584 4801 6505 7680 8352	1304 1104 0863 0608 0359	3 3 3 3 3	
2.0 2.1 2.2 2.3 2.4	2.7257 2.8855 3.0368 31801 3.3162	.0567 .0526 .0487 .0449	1.6401 1.5548 1.4725 1.3958 1.3261	0399 0403 0390 0364 0330	8575 8425 7986 7343 6575	0135 .0054 .0202 .0307 .0372	3 3 3 3	
2.5 2.6 2.7 2.8 2.9	3.4456 3.5693 3.6881 3.8028 3.9141	.0383 .0356 .0333 .0314 .0298	1.2645 1.2112 1.1661 1.1286 1.0982	0291 0250 0210 0173 0140	5747 4914 4116 3381 2726	.0403 .0405 .0387 .0355 .0314	3 3 3 3 3	
3.0 3.1 3.2 3.3 3.4	4.0 2 2 7 4.1 2 9 0 4.2 3 3 7 4.3 3 7 1 4.4 3 9 6	.0286 .0276 .0268 .0263 .0259	10738 10547 10399 10287 10204	0111 0086 0065 0049 0036	2159 1680 1285 0967 0715	.0269 .0225 .0183 .0146 .0114	3 3 3 3	
3.5 3.6 3.7 3.8 3.9	4.5413 46425 47433 48438 49442	.0255 .0253 .0252 .0251 .0250	1.0142 1.0098 1.0066 1.0044 1.0029	0026 0018 0013 0009 0006	0 5 20 0 5 7 2 0 2 6 3 0 1 8 1 0 1 2 4	.0087 .0065 .0048 .0034 .0024	3 3 3 3	
4.0 4.1 4.2 4.3 4.4	5.0 4 4 4 51 4 4 6 5.2 4 4 7 5.3 4 4 8 5.4 4 4 8	.0249 .0249 .0249 .0249	1.0019 1.0012 1.0008 1.0005 1.0003	0004 0002 0001 0001	0083 0055 0036 0023 0014	.0017 .0012 .0008 .0005	3 3 3 3 3	
4.5 4.6 4.7 4.8 4.9	5.5448 5.6448 5.7448 5.8449 5.9449	.0249 .0249 .0249 .0249 .0249	1.0002 1.0001 1.0001 1.0001 1.0000	.0000	0009 0005 0003 0002 0001	.0003 .0002 .0001 .0000	3 3 3 3 3	

5.0

5.0 5 4 3

-.0340

3897

8.0057

69059

TABLE V. - SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR

#### ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72;  $\gamma$ , 1.40] (a) The function IH( $\eta$ ) =  $\int_{-\eta}^{\eta}$  H d $\eta$ 

(a) The function  $IH(\eta) =$ NACA\_ IH17 IH<sub>12</sub> IH<sub>14</sub> IH<sub>13</sub> η  $\Pi_{15}$ N 0 0.0000 0.0000 0.0.00 0.0000 00000 1 -.0.0 1 3 -.00 2 7 -.00 1 0 .0014 .0078 .0223 .0473 .0278 .1111 .2494 .1.23 1 1 1 .0001 2267 .0005 1068 .0025 .0059 4238 .4 4416 1. .0848 .0042 .0192 £859 6571 0394 .6 1359 9792 9357 1 .0061 .7 0660 1.3173 1.2549 .0082 2014 1 1.6087 .8 2813 .0982 1.6949 .0103 1 9. 3751 .0120 .1347 21057 1.9901 1 .1739 1.0 .4818 .0134 2.5422 23914 1 .5999 .7277 .21 41 .25 35 29965 1.1 .0141 2.8043 1 3.2205 1.2 .0141 3.4601 1 .2907 39245 3.6320 1.3 .8631 .0134 1 1.4 1.0041 .0118 .3244 43816 4.0312 .3537 .3780 .3970 1.5 1.1486 .0096 4.8235 4.4117 1 5.2437 5.6366 4.7 6 7 9 5.0 9 5 8 1.2946 .0067 1.6 1 .0034 1.4167 1.7 1 1.8 1.5845 **S000.**-.4111 59980 5.3924 1.9 1.7258 -.0041 .4206 63250 5.6563 1 5.8874 5.0 1.8637 -.0079 .4260 6.6161 1 6.8712 1.9976 -.0116 .4281 2.1 6.0863 1 2.2 2.1272 7.0912 7.2780 .4277 6.2550 -.0151 1 2.2527 -.0183 .4255 63956 1 2.3742 2.4 -.0212 .4221 7.4341 6.5111 2,5 -.0237 7.5625 6.6044 2.4920 .4181 1 2.6 -.0258 -.0276 7.6666 6.6787 6.7370 1 2.6065 4138 2.7181 .4097 7.7 4 9 7 2.8 2.8273 -.0291 4059 7.8149 6.7820 1 29 2.9344 -.0303 4025 7.8654 6.8162 3996 3972 3953 3.0 3.1 3.2 3.0398 7.9040 6.8 4 1 8 -.0313 1 7.9329 6.8607 3.1 4 3 9 -.0320 1 7.9543 6.8745 3.2469 -.0325 1 3,3 3.3492 -.0330 3938 7.9699 6.8844 1 -.0333 3926 7.9811 6.8913 1 3.4 3.4508 3.5519 -.0335 .3917 7.9890 6.8962 1 3.5 .3911 .3906 6.8995 69017 3.6 3.7 3.6527 3.7532 -.0337 -.0338 7.9946 1 7.9984 1 .39 03 .39 01 3.8 3.8 5 3 6 -.0339 8.0009 69032 1 3.9 3.9538 -.0339 8.0026 69042 1 .3899 .3898 .3898 4.0540 8.0038 6.9049 -.0339 4.0 1 69053 4.1 4.1541 -.0340 8.0045 1 4.2 4.2542 -.0340 8.0050 6.9055 1 4.3 4.3542 -.0340 .3897 8.0053 6.9057 1 -.0340 3897 8.0054 6.9058 1 4.4 4.4542 4.5 4.5 5 4 3 -.0340 8.0056 6.9058 .3897 1 -.0340 .3897 4.6 4.6543 8.0056 6.9059 1 4.7 4.7543 -.0340 .3897 8.0057 6.9059 1 4.8 5 4 3 -.0340 .3897 8.0057 4.8 6.9059 1 4.9 4.9543 -.0340 8.0057 3897 6.9059 1

2968

# TABLE V. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR

#### ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; γ, 1.40]

	•	L.	Pr, 0.72; γ, 1.4	n lon	
	·	a) Continued.		Ψ, .	NACA
ŋ	IH <sub>21</sub>	IH <sub>22</sub>	IH <sub>23</sub>	IH <sub>24</sub>	IH <sub>25</sub> N
0 1 2 3 4	0.0000 .0019 .0109 .0316 .0672	0.0000 .0002 .0009 .0025 .0051	0.0000 .0003 .0055 .0310 .0499	0.0000 .0435 .1738 .3899 .6899	0.0000 2 .0509 2 .2033 2 .4559 2 .8055 2
5.6.7.8.9	.1199 .1910 .2808 .3888 .5137	.0086 .0128 .0176 .0224 .0270	.0936 1520 .2237 .3063 .3969	1.0698 1.5239 2.0449 2.6232 3.2481	1.2 4 6 5 1.7 7 1 0 2.3 6 8 5 3.0 2 6 6 3.7 3 1 1
1.0 1.1 1.2 1.3 1.4	.6538 .8067 .9696 1.1397 1.3142	.0312 .0345 .0367 .0379 .0378	.4 9 2 2 .5 8 8 7 .6 8 3 2 .7 7 2 9 .8 5 5 4	3.9 0 7 3 4.5 8 8 1 5.2 7 7 4 5.9 6 2 3 6.6 3 0 8	4.4668 2 5.2180 2 5.9694 2 6.7064 2 7.4161 2
15 16 17 18 19	1.4903 1.6657 1.8383 2.0063 2.1688	.0366 .0344 .0314 .0278	.9290 .9927 10460 10892 11229	7,2721 7,8771 8,4384 8,9508 9,4112	8.0874 8.7115 9.2819 2.7947 10.2484
20 21 22 23 24	2.3249 2.4743 2.6169 2.7531 2.8833	.0196 .0154 .0113 .0075	1.1 481 1.1 661 1.1 780 1.1 852 1.1 889	9,8183 10,1727 10,4764 10,7328 10,9458	10.6433 10.9816 211.2669 211.5039 11.6977
25 26 27 28 29	3.0081 3.1282 3.2441 3.3566 3.4662	.0010 0016 0038 0056 0071	1.1 9 0 0 1.1 8 9 4 1.1 8 7 8 1.1 8 5 7 1.1 8 3 4	11.1202 11.2607 11.3724 11.4597 11.5270	11.8538 11.9776 2 12.0744 2 12.1489 2 12.2054
3.0 3.1 3.2 3.3 3.4	3.5 7 3 5 3.6 7 9 0 3.7 8 3 0 3.8 8 5 9 3.9 8 8 0	0083 0092 0099 0105 0109	1.1812 1.1792 1.1775 1.1761 1.1750	11.5781 11.6164 11.6446 11.6650 11.6797	12.2477 12.2788 12.3014 12.3176 12.3290 2
3.5 3.6 3.7 3.8 3.9	4.0895 4.1905 4.2912 4.3917 4.4920	0111 0114 0115 0116 0117	11741 11734 11729 11726 11723	11.6900 11.6972 11.7021 11.7054 11.7076	123369 123423 2123460 2123484 123500
4.0 4.1 4.2 4.3 4.4	4.5922 4.6924 4.7924 4.8925 4.9925	0117 0118 0118 0118 0118	1.1 722 1.1 720 1.1 720 1.1 719 1.1 719	11.7091 11.7100 11.7106 11.7110 11.7112	123510 123517 123521 2123523 123525 2
4.5 4.6 4.7 4.8 4.9	5.0926 5.1926 5.2926 5.3926 5.4926	0118 0118 0118 0118 0118	11719 11719 11719 11719 11719	11.7114 11.7115 11.7115 11.7115 11.7116	123526 2 123526 2 123527 2 123527 2 123527 2
5.0 5.1 5.2	5.5926 5.6926 5.7926	0118 0118 0118	1.1719 1.1719 1.1719	11.7116 11.7116 11.7116	123527 2 123527 2 123527 2

TABLE V. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; r, 1.40]

(a) Concluded. The function  $IH(\eta) = \int_{0}^{\eta} H d\eta$ 

				·	NACA	
η	IE <sub>31</sub>	IH <sub>32</sub>	IH <sub>33</sub>	IH <sub>34</sub>	IH <sub>35</sub>	N
0 .1 .2 .3 .4	0,0000 .0023 .0135 .0391 .0830	0.0000 .0002 .0013 .0037	0.0 0 0 0 .0 0 2 3 .0 1 5 3 .0 4 5 8 .0 9 7 7	0,0000 0,569 2374 .5101 .9015	0.0000 .0727 .2905 .6510 1.1488	33333
.5 .6 .7 .8	.1475 .2338 .3417 .4703	.0126 .0189 .0259 .0333 .0406	1720 2680 3830 5132 6541	1.3958 1.9848 2.6573 3.4002 4.1984	1.7 7 4 9 2.5 1 6 5 3.3 5 7 4 4.2 7 8 6 5.2 5 9 1	3333
1.0 1.1 1.2 1.3 1.4	.7815 .9586 11458 13397 15369	.0473 .0530 .0576 .0607	.8 01 0 .9 4 9 0 1.0 9 3 6 1.2 3 0 9 1.3 5 7 7	5.0 3 5 8 5.8 9 5 4 6.7 6 0 5 7.6 1 5 0 8.4 4 4 3	6.2769 7.3100 8.3370 9.3387 10.2979	3 3 3 3 3
1.5 1.6 1.7 1.8 1.9	1.7345 1.9296 2.1201 2.3041 2.4806	.0628 .0618 .0597 .0568	14719 15720 16575 17285 17860	9.2354 9.9778 10.6630 11.2857 11.8426	11.2006 12.0357 12.7956 13.4761 14.0758	3 3 3 3 3
2.0 2.1 2.2 2.3 2.4	2.6 4 8 8 2.8 0 8 5 2.9 5 9 8 3.1 0 3 2 3.2 3 9 2	.0493 .0453 .0413 .0375	1.8 31 1 1.8 65 5 1.8 9 0 7 1.9 0 8 6 1.9 2 0 7	12.3330 12.7582 13.1214 13.4268 13.6798	1 4.5 9 6 1 1 5.0 4 0 5 1 5.4 1 4 4 1 5.7 2 4 2 1 5.9 7 7 0	33333
2.5 2.6 2.7 2.8 2.9	3.3 6 8 6 3.4 9 2 3 3.6 1 1 1 3.7 2 5 8 3.8 3 7 1	.0310 .0282 .0259 .0240	1.9 28 4 1.9 32 9 1.9 35 2 1.9 36 0 1.9 35 9	13.8863 14.0523 14.1837 14.2863 14.3652	16.1803 16.3412 16.4669 16.5635 16.6367	3 3 3 3 3
3.0 3.1 3.2 3.3 3.4	3.9 4 5 6 4.0 5 2 0 4.1 5 6 7 4.2 6 0 1 4.3 6 2 6	.0212 .0202 .0195 .0189 .0185	1.9 3 5 3 1.9 3 4 4 1.9 3 3 6 1.9 3 2 8 1.9 3 2 1	1 4.4 2 5 0 1 4.4 6 9 7 1 4.5 0 2 5 1 4.5 2 6 3 1 4.5 4 3 3	16.6914 16.7316 16.7608 16.7816 16.7963	33333
3.5 3.6 3.7 3.8 3.9	4.4 6 4 3 4.5 6 5 5 4.6 6 6 3 4.7 6 6 8 4.8 6 7 2	.0182 .0180 .0178 .0177	1.9 31 5 1.9 31 1 1.9 30 8 1.9 30 5 1.9 30 4	1 4.5 5 5 3 1 4.5 6 3 6 1 4.5 6 9 3 1 4.5 7 3 1 1 4.5 7 5 6	16.8 0 6 6 16.8 135 16.8 182 16.8 214 16.8 234	3 3 3 3 3
4.0 4.1 4.2 4.3 4.4	4.9674 5.0676 5.1677 5.2677 5.3678	.0175 .0175 .0175 .0175	1.9 3 0 3 1.9 3 0 2 1.9 3 0 1 1.9 3 0 1 1.9,3 0 1	1 4.5 7 7 3 1 4.5 7 8 4 1 4.5 7 9 0 1 4.5 7 9 5 1 4.5 7 9 7	16.8247 16.8256 16.8261 16.8264 16.8266	3 3 3 3 3
4.5 4.6 4.7 4.8 4.9	5.4 6 7 8 5.5 6 7 8 5.6 6 7 8 5.7 6 7 8 5.8 6 7 8	.0174 .0174 .0174 .0174	1.9 3 0 1 1.9 3 0 1 1.9 3 0 1 1.9 3 0 1 1.9 3 0 1	1 4.5 7 9 9 1 4.5 8 0 0 1 4.5 8 0 1 1 4.5 8 0 1 1 4.5 8 0 1	16.8268 16.8268 16.8269 16.8269 16.8269	33333
5.0 5.1 5.2 5.3	5.9678 6.0678 6.1678 6.2678	.0174 .0174 .0174 .0174	1.9 3 0 1 1.9 3 0 1 1.9 3 0 1 1.9 3 0 1	1 4.5 8 0 1 1 4.5 8 0 1 1 4.5 8 0 1 1 4.5 8 0 1	16.8269 16.8269 16.8269 16.8269	3 3 3 3

### TABLE V. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR

### ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; \u00e3, 1.40]

		(ъ) Т	ne function H(	η)	NA.	CA
η	H <sub>ll</sub>	H <sub>12</sub>	H <sub>13</sub>	H <sub>14</sub>	H <sub>15</sub>	N
0 1 2 3 .4	0.0 0 0 0 .0 3 3 3 .1 0 0 0 .1 9 4 2 .3 1 0 0	0.0000 .0020 .0057 .0102	0.0000 0198 0028 .0404 .1000	0.0000 .5556 1.1091 1.6554 2.1869	00000 .5343 10659 15884 20923	11111
.5 .6 .7 .8 .9	.4414 .5825 .7272 .8696 1.0044	.0183 .0206 .0210 .0194 .0159	.1674 .2350 .2962 .3462 .3813	2.6 9 3 8 3.1 6 4 8 3.5 8 8 5 3.9 5 3 4 4.2 4 9 3	2.5 6 6 4 2.9 9 8 3 3.3 7 6 0 3.6 8 8 4 3.9 2 6 7	11111
1.0 1.1 1.2 1.3 1.4	1.1 2 6 8 1.2 3 2 8 1.3 1 9 7 1.3 8 5 7 1.4 3 0 6	.0106 .0039 0037 0115 0190	.3997 .4008 .3856 .3563 .3160	4.4679 4.6038 4.6543 4.6205 4.5068	4.0 8 4 6 4.1 5 9 3 4.1 5 1 6 4.0 6 5 5 3.9 0 8 8	1 1 1 1
1.5 1.6 1.7 1.8 1.9	1.4552 1.4613 1.4516 1.4291 1.3973	0257 0312 0351 0376 0384	.2682 .2168 .1653 .1167	4.3211 4.0738 3.7773 3.4454 3.0920	3.6 9 1 6 3.4 2 6 2 3.1 2 5 9 2.8 0 4 3 2.4 7 4 3	1 1 1 1 1
2.0 2.1 2.2 2.3 2.4	13592 13178 12755 12344 11958	0379 0362 0336 0305 0269	.0367 .0075 0142 0291 0380	27305 23731 2.0301 1.7096 1.4174	2.1478 1.8344 1.5419 1.2756 1.0390	111111
2.5 2.6 2.7 2.8 2.9	1.1 6 0 8 1.1 2 9 9 1.1 0 3 2 1.0 8 0 7 1.0 6 2 1	0233 0197 0163 0133 0106	0420 0423 0401 0362 0314	11570 9301 .7363 .5740 .4408	.8332 .6580 .5117 .3920 .2958	11111
3.0 3.1 3.2 3.3 3.4	1.0 4 7 1 1.0 3 5 2 1.0 2 5 9 1.0 1 8 8 1.0 1 3 4	0083 0064 0049 0036 0026	0264 0216 0172 0134 0101	3335 2485 1824 1320 0940	.2199 .1611 .1163 .0827 .0579	111111
3.5 3.6 3.7 3.8 3.9	1.0 0 9 4 1.0 0 6 6 1.0 0 4 5 1.0 0 3 0 1.0 0 2 0	0019 0013 0009 0006 0004	0075 0055 0039 0027 0018	.0660 .0457 .0311 .0209	.0400 .0272 .0183 .0121	1 1 1 1 1
4.0 4.1 4.2 4.3 4.4	1.0013 1.0008 1.0005 1.0003 1.0002	0003 0002 0001 0001	0012 0007 0004 0002 0001	.0090 .0058 .0037 .0023	.0051 .0032 .0020 .0012 .0007	1 1 1 1 1
4.5 4.6 4.7 4.8 4.9	1.0001 1.0001 1.0001 1.0000 1.0000	.0000	.0000 .0000 .0000	.0009 .0005 .0003 .0002	.0004 .0003 .0001 .0001	11111
5.0	10000	.0000	.0000	.0000	.0000	1

### TABLE V. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR

#### ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; Y, 1.40]

(ъ)	Continued	The function	H(n)
(0)	COntinued	THE TUNCTION	п. и.

		(b) Contin	ued. The funct	ion H(η)	NA.	CA
η	H <sub>21</sub>	H <sub>22</sub>	H <sub>23</sub>	H <sub>24</sub>	н <sub>25</sub>	N
0 .1 .2 .3 .4	0.0 0 0 0 .0 4 6 5 .1 4 2 4 .2 7 6 6 .4 3 8 4	0.0000 .0039 .0115 .0210 .0306	0.0000 .0170 .0969 .2183 .3622	0.0000 .8695 17347 25854 3.4069	0.0000 1.0177 2.0287 3.0179 3.9641	00000000000000000000000000000000000000
.5 .6 .7 .8	.6176 .8046 .9905 1.1676 1.3288	.0391 .0452 .0483 .0480	.5118 .6534 .7760 .8716 .9352	4.1813 4.8898 5.5134 6.0353 6.4415	4.8429 5.6294 6.3007 6.8378 7.2269	& & & & & & & & & & & & & & & & & & &
1.0 1.1 1.2 1.3 1.4	1.4688 1.5833 1.6699 1.7278 1.7575	.0375 .0281 .0171 .0054	.9646 .96055 .9255 .8642 .7827	6.7222 6.8722 6.8917 6.7862 6.5658	7.4606 7.5379 7.4650 7.2538 6.9215	2222
1.5 1.6 1.7 1.8 1.9	1.7614 1.7427 1.7055 1.6543 1.5936	0170 0263 0335 0386 0415	.6874 .5851 .4819 .3831 .2928	6.2451 5.8418 5.3758 4.8677 4.3379	6.4894 5.9810 5.4210 4.8333 4.2402	2222
2.0 2.1 2.2 2.3 2.4	1.5276 1.4533 13937 13312 12740	0424 0416 0394 0362 0324	.2137 .1472 .0937 .0524 .0222	3.8 0 5 2 3.2 8 6 1 2.7 9 4 1 2.3 3 9 4 1.9 2 9 0	3.6610 3.1114 2.6033 2.1448 1.7402	00000000000000000000000000000000000000
2.5 2.6 2.7 2.8 2.9	12230 11787 11409 11095 10838	0283 0242 0202 0165 0133	.0014 0118 0192 0224 0226	1.5665 1.2531 .9874 .7664 .5861	1.3907 1.0947 .8489 .6486	& & & & & &
3.0 3.1 3.2 3.3 3.4	1.0 6 3 1 1.0 4 6 9 1.0 3 4 3 1.0 2 4 7 1.0 1 7 6	0105 0081 0062 0046 0034	0211 0185 0156 0127 0100	.4416 .3278 .2398 .1729 .1228	.3621 .2647 .1907 .1354 .0947	& & & & & &
3.5 3.6 3.7 3.8 3.9	1.0123 1.0085 1.0058 1.0039 1.0026	0025 0017 0012 0009 0006	0076 0057 0042 0030 0021	.0859 .0592 .0402 .0269 .0178	.0653 .0444 .0297 .0196 .0128	2 2 2 2
4.0 4.1 4.2 4.3 4.4	1.0 0 1 7 1.0 0 1 1 1.0 0 0 7 1.0 0 0 4 1.0 0 0 3	0004 0003 0002 0001 0001	0014 0010 0006 0004 0003	.0116 .0074 .0047 .0029 .0018	.0082 .0052 .0032 .0020 .0012	2 2 2 2 2
4.5 4.6 4.7 4.8 4.9	1.0002 1.0001 1.0001 1.0001 1.0000	0001 0001 .0000 .0000	0002 0002 0001 0001	.0011 .0007 .0004 .0002	.0007 .0004 .0002 .0001 .0001	8 8 8 8
5.0 5.1 5.2 5.3	1.0000 1.0000 1.0000	.0000 .0000 .0000	.0000 .0000 .0000	.0001 .0001 .0001	.0001 .0001 .0001	2 2 2 2 2

TABLE V. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR

#### ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; Y, 1.40]

(b) Concluded. The function  $H(\eta)$ 

					- Territ	
η	H <sub>31</sub>	H <sub>32</sub>	H <sub>33</sub>	H <sub>34</sub>	H <sub>35</sub>	' N
0 .1 .2 .3 .4	0.0 0 0 0 .0 4 7 3 .1 7 6 7 .3 4 2 3 .5 3 9 0	0.0000 .0056 .0167 .0306 .0449	0.0 0 0 0 .0 6 2 4 .2 0 9 0 .4 0 7 9 .6 3 1 0	0.0 0 0 0 1.1 3 8 3 2.2 6 9 5 3.3 7 7 8 4.4 4 0 7	0.0 0 0 0 1.4 5 4 5 2.8 9 7 3 4.3 0 3 3 5.6 3 7 8	3 3 3 3 3
.5 .6 .7 .8	.7529 .9719 11853 13840 1.5607	.0576 .0673 .0729 .0739 .0705	.8547 1.0601 1.2330 1.3638 1.4473	5.4327 6.3275 7.1008 7.7320 8.2059	6.8629 7.9423 8.8443 9.5447 10.0285	3 3 3 3 3
1.0 1.1 1.2 1.3 1.4	1.7097 1.8271 1.9109 1.9609 1.9788	.0629 .0520 .0388 .0244 .0099	1.4819 1.4697 1.4153 1.3255 1.2084	8.5 1 3 1 8.6 5 1 1 8.6 2 4 0 8.4 4 2 2 8.1 2 1 7	10,2903 10,3346 10,1744 9,8307 9,3302	3 3 3 3 3
1.5 1.6 1.7 1.8 1.9	1.9675 1.9313 1.8751 1.8043 1.7242	0037 0157 0255 0328 0376	1.0730 .9280 .7816 .6406 .5104	7.6829 7.1496 6.5470 5.9009 5.2359	8.7 0 4 0 7.9 8 5 0 7.2 0 6 7 6.4 0 0 7 5.5 9 5 7	3 3 3 3 3
2.0 2.1 2.2 2.3 2.4	1.6395 1.5544 1.4723 1.3956 1.3260	0400 0403 0390 0364 0330	.3947 .2954 .2132 .1475 .0968	4.5 7 4 4 3.9 3 5 5 3.3 3 4 4 2.7 8 2 7 2.2 8 7 4	4.8162 4.0818 3.4068 2.8007 2.2679	3 3 3 3 3
2.5 2.6 2.7 2.8 2.9	1.2644 1.2111 1.1660 1.1286 1.0982	0291 0250 0211 0173 0140	.0591 .0323 .0141 .0026	1,8523 1,4776 1,1614 .8994 .6862	1.8092 1.4220 1.1012 .8403 .6319	3 3 3 3
3.0 3.1 3.2 3.3 3.4	1.0738 1.0547 1.0399 1.0287 1.0204	0111 0086 0066 0049 0036	0075 0086 0084 0075 0062	.5160 .3823 .2791 .2008 .1424	.4683 .3420 .2462 .1747 .1221	3 3 3 3
3.5 3.6 3.7 3.8 3.9	1.0 1 4 2 1.0 0 9 8 1.0 0 6 6 1.0 0 4 4 1.0 0 2 9	0026 0019 0013 0009 0006	0050 0038 0028 0020	.0994 .0684 .0464 .0310	.0841 .0571 .0382 .0252 .0164	3 3 3 3 3
4.0 4.1 4.2 4.3 4.4	1.0019 1.0012 1.0008 1.0005 1.0003	0005 0003 0002 0002	0009 0006 0004 0002	.0133 .0085 .0054 .0033	.0105 .0066 .0041 .0025 .0015	3 3 3 3 3
4.5 4.6 4.7 4.8 4.9	1.0002 1.0001 1.0001 1.0001 1.0000	0001 .0000 .0000 .0000	.0000 .0000 .0000 .0000	.0012 .0007 .0004 .0003 .0002	.0009 .0005 .0003 .0002	3 3 3 3 3
5.0 5.1 5.2 5.3	1.0000 1.0000 1.0000 1.0000	.0000	.0 0 0 0 .0 0 0 0 .0 0 0 0	.0001 .0001 .0001	.0000	3 3 3 3

TABLE V. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; γ, 1.40]

(c) The function  $H'(\eta)$ 

	(c) The function H'(η)			NA	سر ۸۸	
η	Hiı	H <sub>1</sub> 2	H'13	H <sub>14</sub>	H <sub>15</sub>	N
0 •1 •2 •3 •4	0.1 4 7 9 ·50 9 6 ·8 1 3 8 1.0 5 9 8 1.2 4 6 4	0.0082 .0302 .0427 .0462 .0416	-0.4201 .0042 .3179 .5298 .6492	5.5 5 7 4 5.5 5 1 2 5.5 1 0 1 5.4 0 4 0 5.2 0 9 3	5.3452 5.3370 5.2837 5.1497 4.9101	1 1 1 1 1 1
.5 .6 .7 .8	1.3728 1.4386 1.4450 1.3949 1.2932	.0303 .0139 0055 0259 0450	.6865 .6534 .5632 .4301 .2695	4.9087 4.4931 3.9610 3.3197 2.5847	4.5 5 0 6 4.0 6 7 5 3.4 6 7 5 2.7 6 6 5 1.9 8 8 8	1 1 1 1
1.0 1.1 1.2 1.3 1.4	1.1 4 7 7 .9 6 8 2 .7 6 6 2 .5 5 4 3 .3 4 5 0	0609 0722 0778 0775 0717	.0968 0731 2268 3538 4467	1.7794 .9329 .0789 7475 -1.5121	1.1652 .3306 4790 -12289 -1.8888	1 1 1 1
1.5 1.6 1.7 1.8 1.9	.1 495 0 230 -1663 2771 3550	0613 0476 0321 0163 0014	5020 5203 5051 4630 4018	-2.1847 -2.7412 -3.1653 -3.4497 -3.5959	-2.4349 -2.8515 -3.1320 -3.2784 -3.3005	111111
2.0 2.1 2.2 2.3 2.4	4020 4220 4196 4003 3690	.0115 .0218 .0293 .0341 .0363	3296 2542 1817 1169 0625	-3.6132 -3.5176 -3.3294 -3.0715 -2.7669	-3,2145 -3,0405 -2,8009 -2,5181 -2,2129	1 1 1 1
2.5 2.6 2.7 2.8 2.9	3303 2882 2456 2048 1673	.0364 .0348 .0321 .0287 .0248	0 1 9 8 .0 1 1 4 .0 5 2 3 .0 4 4 4 .0 4 9 6	-2.4374 -2.1021 -1.7765 -1.4722 -1.1972	-1.9 0 3 3 -1.6 0 3 8 -1.3 2 5 1 -1.0 7 4 3 8 5 5 0	1 1 1 1
3.0 3.1 3.2 3.3 3.4	1340 1054 0814 0617 0458	.0210 .0173 .0139 .0110	.0497 .0465 .0412 .0353 .0291	9558 7495 5775 4373 3256	6684 5135 3878 -2879 -2102	1 1 1 1
3.5 3.6 3.7 3.8 3.9	0338 0241 0175 0122 0083	.0064 .0048 .0035 .0025	.0233 .0181 .0137 .0102	2384 1716 1217 0844 0580	1 5 1 1 1 0 6 7 0 7 4 3 0 5 0 6 0 3 4 2	1 1 1 1
4.0 4.1 4.2 4.3 4.4	0 0 5 6 0 0 3 7 0 0 2 4 0 0 1 5 0 0 1 0	.0012 .0008 .0006 .0004 .0002	.0053 .0037 .0025 .0017	0392 0261 0169 0110 0069	0227 0149 0095 0061 0038	1 1 1 1
4.5 4.6 4.7 4.8 4.9	0 0 0 6 0 0 0 4 0 0 0 2 0 0 0 1	.0002 .0001 .0001 .0000	.0008 .0005 .0003 .0001	0044 0026 0016 0009 0005	0023 0014 0009 0005 0003	1 1 1 1
5.0 5.1 5.2 5.3	.0 0 0 0 .0 0 0 0 .0 0 0 0	.0000 .0000 .0000	.0000 .0000 .0000	-0003 -0002 -0001 0000	0002 0001 .0000	1 1 1 1

TABLE V. - Continued. SCLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; \u00e7, 1.40]

r		(d) Cont	inued. The functi	on H'(η)	- KA	ČA.
η	H'21	H'22	H! 25	H <sub>24</sub>	H.'.	N
0 .1 .2 .3 .4	01802 .7308 11688 1.4974 1.7214	0.0145 .0608 .0882 .0981 .0926	-0.2200 .5214 1.0403 1.3564 1.4932	8.6985 8.6855 8.6008 8.3886 8.0108	10,1820 10,1618 10,0325 9,7161 9,2336	88888
.5 .6 .7 .8	1.8 4 6 2 1.8 7 8 7 1.8 2 7 3 1.7 0 1 9 1.5 1 4 2	.0745 .0472 .0143 0205 0536	1.4771 1.3366 1.1015 .8017 .4665	7.4464 6.6905 5.7539 4.6607 3.4470	8.3668 7.3248 6.0700 4.6490 3.1207	88888
1.0 1.1 1.2 1.3 1.4	1,2780 1,0086 .7223 .4356 1637	0819 1030 1154 1186 1133	.1237 2021 4901 7244 8953	2.1577 .8433 4432 -16506 -27328	1.5507 .0071 -1.4456 -2.7499 -3.8598	88888
1.5 1.6 1.7 1.8 1.9	0803 2868 4498 5671 6404	1008 0829 0618 0398 0187	9989 -1.0374 -1.0179 9513 8506	-3.6519. -4.3809 -4.9051 -5.2224 -5.3422	-4.7432 -5,3830 -5.7775 -5.9390 -5.8915	88888
2.0 2.1 2.2 2.3 2.4	6738 6735 6467 6007 5421	.0001 .0157 .0275 .0355	7292 5996 4722 3547 2522	-5.2842 -5.0754 -4.7476 -4.3343 -3.8683	-5.6674 -5.3047 -4.8427 -4.3198 -3.7703	88888
2.5 2.6 2.7 2.8 2.9	4770 4102 3453 2848 2305	.0417 .0409 .0383 .0347 .0304	1673 1003 0503 0150 .0081	-33791 -28920 -24271 -19985 -16157	-3.2 23 4 -2.7 01 9 -2.2 21 9 -1.7 9 3 9 -1.4 2 2 5	& & & & & & & & & & & & & & & & & & &
3.0 3.1 3.2 3.3 3.4	1832 1430 1097 0827 0613	0258 0215 0174 0138 0107	.0216 .0282 .0298 .0284 .0253	-12829 -10009 -7675 -5787 -4292	-1.1084 8490 6394 4737 3452	0 0 0 0 0 0
3.5 3.6 3.7 3.8 3.9	0448 0321 0227 0157 0107	.0081 .0060 .0044 .0031	.0214 .0174 .0136 .0103 .0076	3131 2245 1587 1096 0750	2476 1746 1214 0826 0556	8 8 8 8
4.0 4.1 4.2 4.3 4.4	-0072 -0048 -0031 -0020 -0012	0015 0010 0007 0004 0002	.0056 .0039 .0027 .0018 .0012	0506 0336 0217 0140 0088	0369 0342 0154 0098 0061	2222
4.5 4.6 4.7 4.8 4.9	-0008 -0005 -0003 -0002 -0001	0001 0001 0000 0000	.0008 .0005 .0003 .0001	-0055 -0034 -0021 -0012 -0007	0038 0023 0014 0008 0004	2222
5.0 5.1 5.2 5.3	0000 0000 0000 0000	0000 0000 0000 0000	.0 0 0 0 .0 0 0 0 .0 0 0 0	0003 0002 0001 0000	0002 0001 0001	2 2 2 2 2

TABLE V. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR

## ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; Y, 1.40]

(0)	Concluded.	The	function	ות)ים

	(c) Concluded. The function H'(η)					
η	H31	H;32	H;33	H;4	H <sub>35</sub>	N
0 19354	0.2 0 4 2 .9 1 1 3 1.4 5 1 8 1.8 3 5 3 2.0 7 4 5	0.0195 .0884 .1292 .1446 .1380	0.0899 1.1008 1.7774 2.1532 2.2698	1 1.3 8 7 9 1 1.3 6 6 7 1 1.2 3 0 6 1 0.8 9 7 5 1 0.3 1 8 8	14.5535 14.5183 14.2961 13.7641 12.8617	พพพพพ
.5 .6 .7 .8 .9	2.1837 2.1783 2.0743 1.8879 1.6362	.1139 .0774 .0336 0125 0562	21731 19111 15308 10768 5900	9.4767 8.3784 7.0529 5.5460 3.9150	11.5807 9.9538 8.0448 5.9380 3.7290	33333
1.0 1.1 1.2 1.3 1.4	1.3367 1.0079 .6682 .3357	0937 1222 1399 1463 1422	.1064 3431 7340 -10489 -12773	2.22 47 .5 4 2 3 -1.0 6 6 6 -2.5 4 2 1 -3.8 3 3 6	1.5156 6085 -2.5599 -42699 -5.6878	33333
1.5 1.6 1.7 1.8 1.9	2454 4707 6437 7632 8318	1292 1095 0857 0603 0356	-1.4162 -1.4697 -1.4474 -1.3635 -1.2344	-4.9022 -5.7230 -6.2858 -6.5948 -6.6674	-6.7821 -7.5412 -7.9725 -8.0995 -7.9594	33333
2.0 2.1 2.2 2.3 2.4	8552 8409 7976 7336 6570	0132 .0056 .0203 .0307 .0375	-1.0770 9070 7377 5792 4383	-6.5 3 1 3 -6.2 2 2 1 -5.7 7 9 4 -5.2 4 4 1 -4.6 5 5 1	-7.5 9 8 0 -7.06 6 6 -6.4 1 7 1 -5.6 9 8 7 -4.9 5 5 2	33333
2.5 2.6 2.7 2.8 2.9	5743 4911 4115 3380 2726	.0403 .0405 .0387 .0355	3187 2214 1454 0887 0481	-4.0470 -3.4489 -28833 -23660 -1.9066	-4.2 2 2 9 -3.5 3 0 1 -2.8 9 6 3 -2.3 3 3 7 -1.8 4 7 4	3 3 3 3 3
3.0 3.1 3.2 3.3 3.4	2159 1680 1285 0967 0716	.0269 .0225 .0183 .0146	0208 0034 .0065 .0114 .0130	-15094 -11745 8984 6759 5003	-1.4374 -1.0996 8272 6123 4460	33333
3.5 3.6 3.7 3.8 3.9	0422 0373 0263 0182 0124	.0086 .0064 .0047 .0033	.0124 .0109 .0090 .0070	3642 2607 1839 1267 0865	3196 2253 1566 1064 0715	3 3 3 3 3 3
4.0 4.1 4.2 4.3 4.4	0083 0055 0035 0023 0014	.0016 .0011 .0007 .0004 .0002	.0039 .0028 .0019 .0013	0583 0387 0249 0160 0101	0476 0311 0198 0125 0078	3 3 3 3 3 3
4.5 4.6 4.7 4.8 4.9	0009 0005 0003 0002 0001	.0002 .0002 .0001 .0001	.0006 .0004 .0002 .0001	~.0063 ~.0038 ~.0023 ~.0013 ~.0009	0048 0029 0018 0010 0006	33333
5.0 5.1 5.2 5.3	.0000 .0000 .0000	.0000 .0000 .0000 .0000	.0000 .0000 .0000	0006 0004 0002 .0001	0003 0001 .0000 .0000	3 3 3 3

TABLE VI. - ASYMPTOTIC VALUES

APPEARING IN EXPRESSION FOR

DISPLACEMENT THICKNESS

[Pr, 0.72; γ, 1.4] NACA								
	N = 1	N = 2	N = 3					
α <sub>Nl</sub>	-4.4764	-6.5114	-7.9898					
αNS	.0126	1230	2688					
α <sub>N3</sub>	-5.1946	-8.3220	-10.6824					
β <sub>Nl</sub>	-2.0346	-3.0786	-3.8104					
$\beta_{\rm N2}$	.0566	.0096	0476					
B <sub>Nl</sub>	-1.8294	-2.9060	-3.6564					
B <sub>N2</sub>	.0680	.0236	0348					
B <sub>N3</sub>	7794	-2.3438	-3.8602					
B <sub>N4</sub>	-16.0112	-23.4228	-29.1598					
B <sub>N5</sub>	-13.8116	-24.7052	-33.6536					

TABLE VII. - SOLUTION OF FIRST ORDER MOMENTUM EQUATION

[Pr, 1; γ, 1.40]

η	g <sub>12</sub>	g <sub>12</sub>	g" <sub>12</sub>	η	g <sub>12</sub>	<b>s</b> i2	g <sub>12</sub>	N
0 1.2 3 4	0.0000 .0016 .0060 .0123 .0198	0.0000 .0312 .0544 .0701 .0787	0.3516 2719 .1941 .1203 .0521	00 200 00 00 00 00 00 00 00 00 00 00 00	0.0232 .0209 .0199 .0191 .0186	-0.0150 0115 0087 0063 0045	0.0374 .0318 .0260 .0206 .0158	111111
.5 .6 .7 .8	.0278 .0357 .0431 .0494 .0545	.0808 .0775 .0692 .0575	0085 0599 1010 1508 1489	3.0 3.1 3.2 3.3 3.4	.0182 .0180 .0178 .0177	0032 0022 0014 0009 0006	.0117 .0085 .0060 .0041 .0027	1 1 1 1
1.0 1.1 1.2 1.3 1.4	.0581 .0601 .0606 .0598	.0281 .0126 0019 0143 0254	1555 1516 1385 1183 0933	3.5 3.6 3.7 3.8 5.9	.0175 .0175 .0175 .0175	0004 0002 0001 0001	.0 0 1 8 .0 0 1 1. .0 0 0 7 .0 0 0 4	1 1 1 1
1.5 1.6 1.7 1.8 1.9	.0548 .0512 .0472 .0430 .0389	0334 0356 0412 0414 0398	0660 0387 0135 .0080 .0849	4.0 4.1 4.2 4.3 4.4	.0175 .0175 .0175 .0175	.0000	.0002 .0001 .0000 .0000	1 1 1 1
2.0 2.1 2.2 2.3 2.4	.0351 .0316 .0286 .0260 .0239	0366 0326 0280 0234 0190	.0367 .0437 .0464 .0456 .0423	4,5	.0175	.0000	.0000	1 1 1 1 1

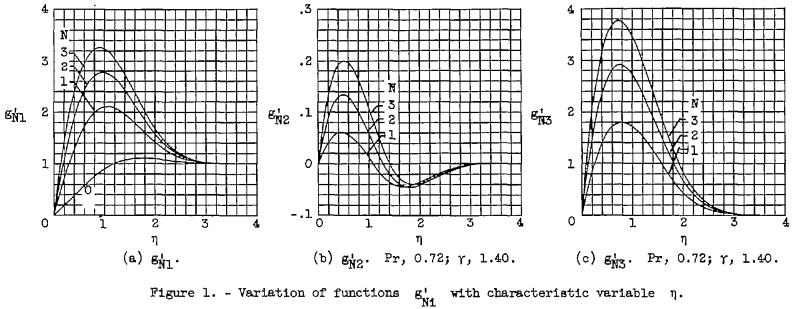


Figure 2. - Variation of functions  $~h_{\mbox{Ni}}~$  with characteristic variable  $~\eta.$  Pr, 0.72;  $\gamma,$  1.40.

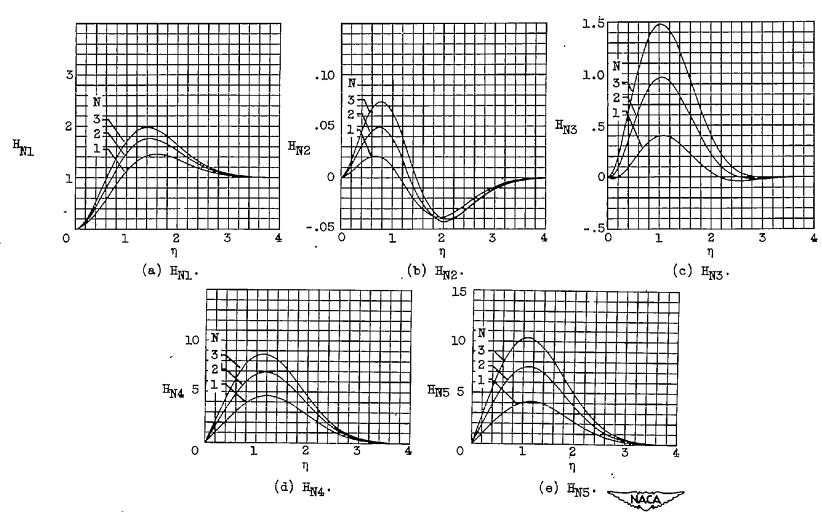
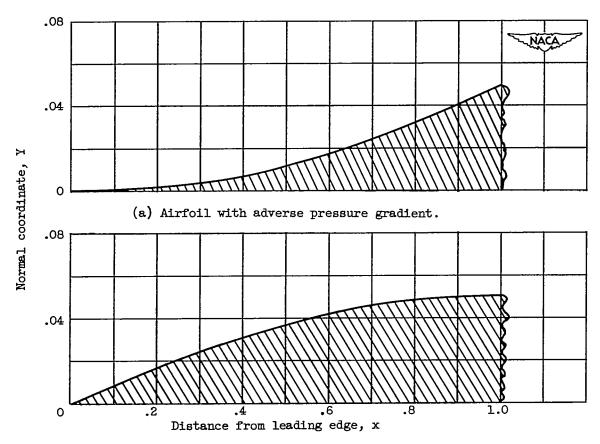


Figure 3. - Variation of functions  $H_{Ni}$  with characteristic variable  $\eta$ . Pr, 0.72;  $\gamma$ , 1.40.



(b) Airfoil with favorable pressure gradient.

Figure 4. - Airfoil shapes used in examples.

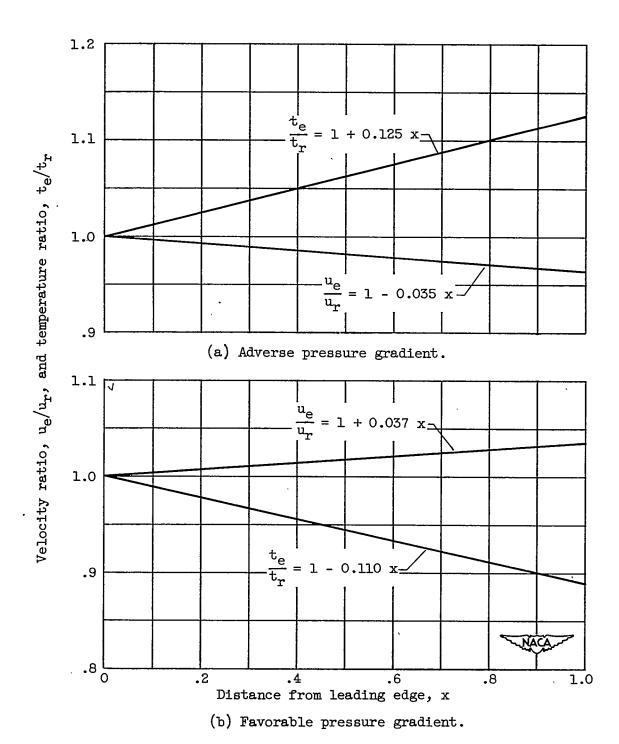
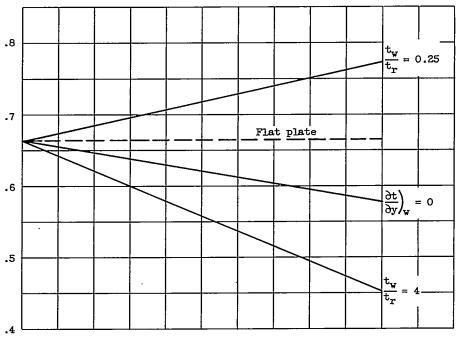
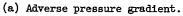
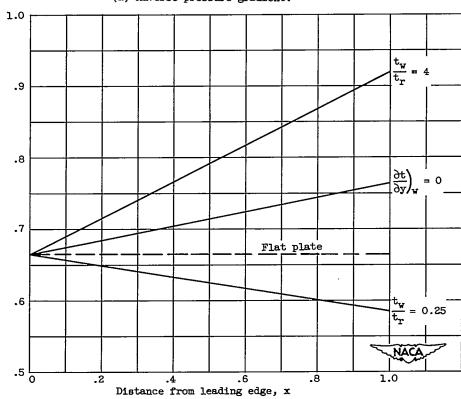


Figure 5. - External velocity and temperature distributions on airfoils used for examples.  $M_{\infty}$ , 3.







(b) Favorable pressure gradient.

Figure 6. - Local skin friction as a function of distance from leading edge.  $M_{\!_{\mathbf{m}}},\ 3.$ 



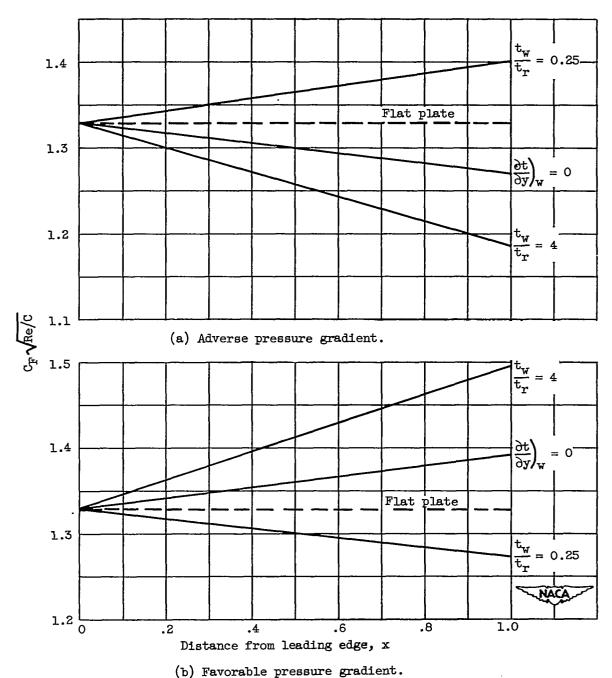


Figure 7. - Average friction drag as a function of distance from leading edge.  $M_{\infty}$ , 3.

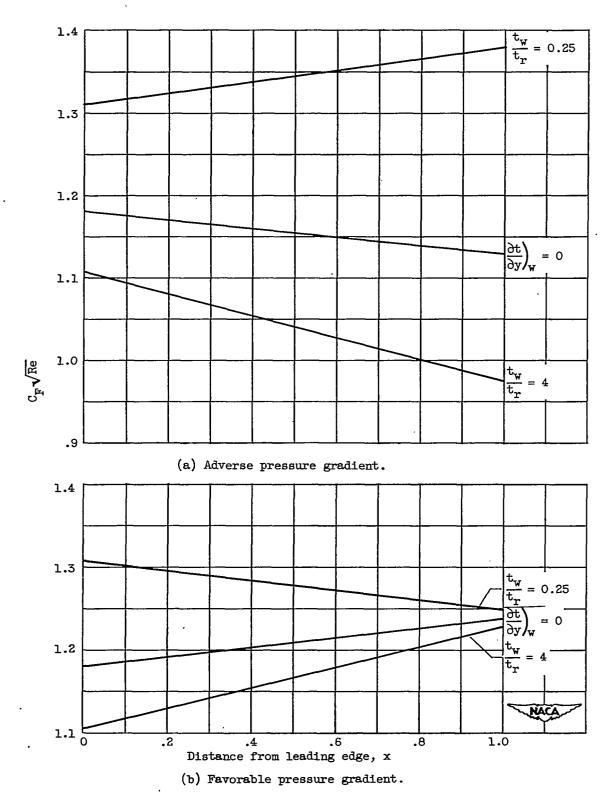


Figure 8. - Average friction drag coefficient as a function of distance from leading edge.  $M_{\bullet}$ , 3;  $t_{\infty}$ , -67° F.

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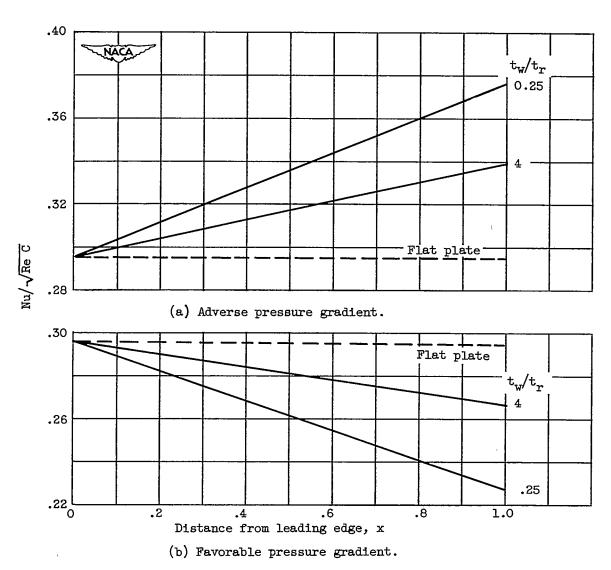


Figure 9. - Heat transfer as a function of distance from leading edge.  $M_{\varpi},\ 3.$ 

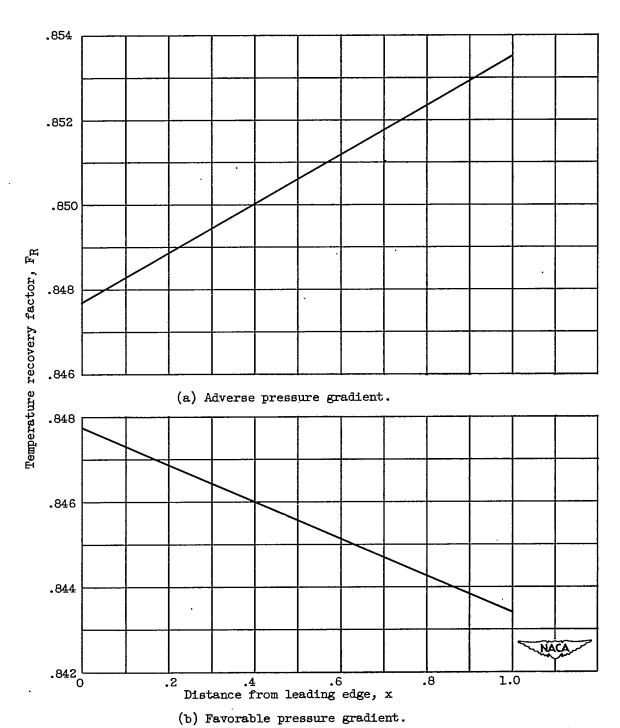


Figure 10. - Temperature recovery factor as a function of distance from leading edge.  $\rm M_{\infty},\ 3.$ 



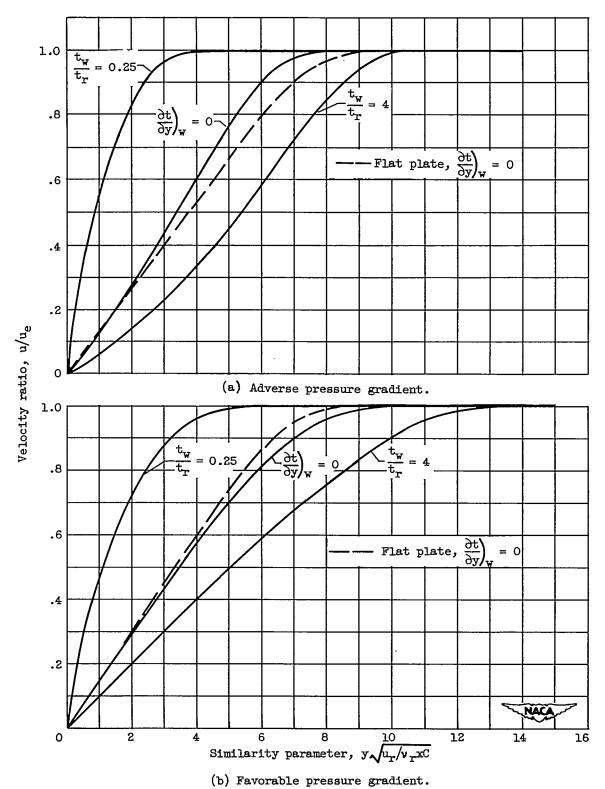


Figure 11. - Velocity profiles.  $M_{\infty}$ , 3; x, 1.

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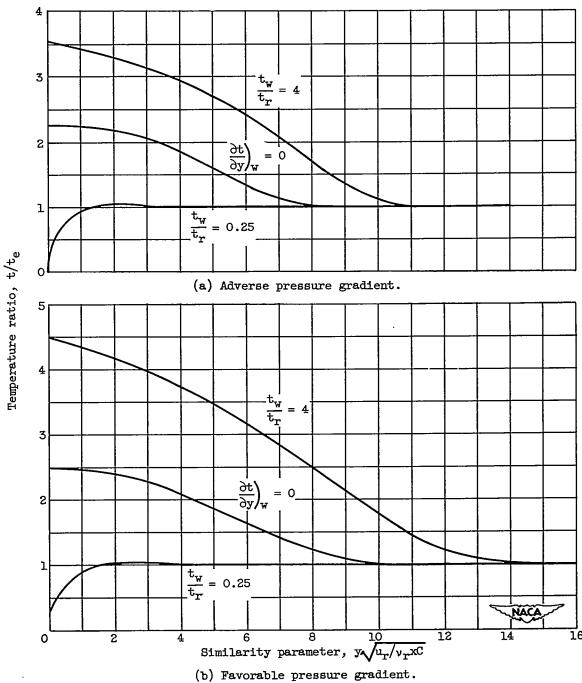
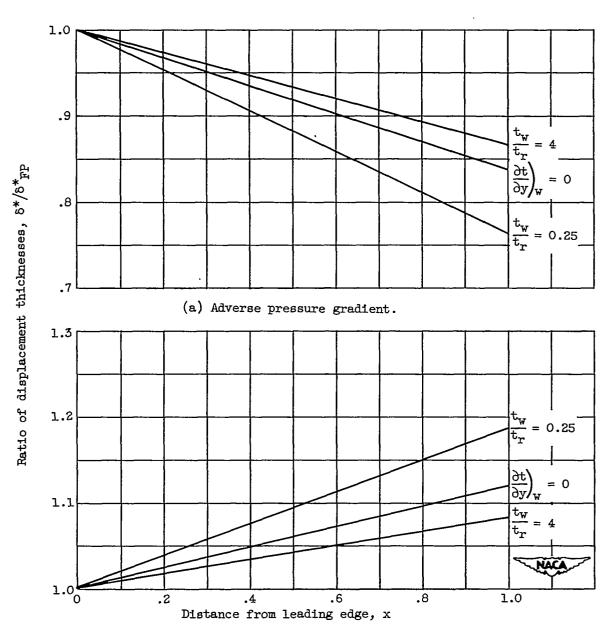


Figure 12. - Temperature profiles.  $M_{\infty}$ , 3; x, 1.

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(b) Favorable pressure gradient.

Figure 13. - Ratio of displacement thicknesses as a function of distance from leading edge.  $\rm M_{\infty}$ , 3.

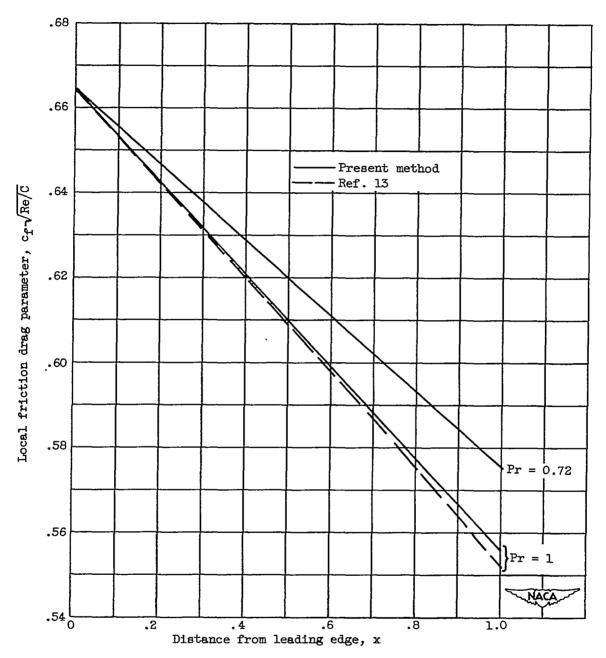


Figure 14. - Comparison of present results with results of reference 13. Zero heat transfer; adverse pressure gradient;  $\rm M_{\infty},\ 3.$